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AN ATTEMPT TO CORRELATE NORMAL VORTICITIES WITH
TOTAL PRESSURE DISTORTION PATTERNS AT THE ENTRANCE
TO A GAS TURBINE ENGINE

PERFORMANCE/STABILITY DIVISION
DIRECTORATE OF ENGINEERING AND TEST
DEPUTY FOR PROPULSION

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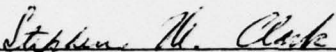
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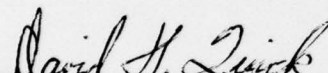
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distortion between steady-state and worst-case pressure measurements. Probe-by-probe analysis of the data indicates that a correlation may exist between vorticity and the distortion increase. Although a final prediction technique is not developed, an improved test program and refined data analysis techniques are outlined.

Abstract

An attempt is made to predict the worst-case pressure distortion produced by a distortion screen at the entrance to a gas turbine engine using only steady-state total pressure measurements. The vector sum of radial and circumferential vorticity, called normal vorticity, is compared to both the difference between steady-state and worst-case distortion patterns, and the standard deviation of the high response pressure measurements, called "turbulence." Average values of "turbulence" and vorticity are found to be unrelated to the increase in distortion between steady-state and worst-case pressure measurements. Probe-by-probe analysis of the data indicates that a correlation may exist between vorticity and the distortion increase. Although a final prediction technique is not developed, an improved test program and refined data analysis techniques are outlined.

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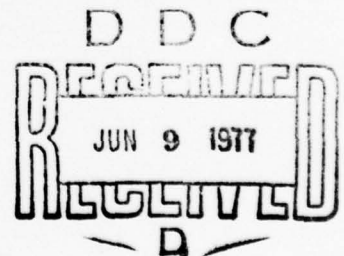


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Chapter I Introduction

Background

Most turbine engines for modern aircraft are designed to operate with the air entering the engine in a uniform velocity profile. A few engines are designed to have the inlet velocity lower at the tip of the compressor than at the hub to optimize the aircraft system performance. In both cases, engine inlet distortion is defined as any perturbation in the compressor inlet velocity profile from the desired velocity profile. If the inlet distortion is too great, the compressor will stall.

Due to the difficulty of measuring the velocity profile of the air entering a turbine engine, indirect measurements must be taken. Current techniques for approximating the inlet velocity variation involve the use of an array of total pressure probes at a turbine engine inlet. Engine inlet velocities are on the order of 0.5 Mach number and incompressible fluid flow is assumed at the measurement plane. The static pressure is nearly uniform if the instrumentation plane is placed far enough in front of the compressor face and its flow redistributions. Therefore, the variation of the total pressure of the air entering a turbine engine is a reasonable approximation of the variation of the inlet velocity. The total pressure probes are usually designed so that good total pressure recovery can be obtained for engine inlet air velocity varying up to 30° from the axial direction.

Current Practice

The development of the F-111 aircraft system was plagued

with engine/inlet aerodynamic compatibility problems. Most of the problems were caused by the aircraft inlet producing more flow distortion than the engine could tolerate. Since the F-111 system development, the U.S. Air Force has directed that during the development of new aircraft systems, especially supersonic aircraft systems, the inlet pressure distortion be measured with high response instrumentation both during scale model tests and flight tests, and that turbine engine qualification tests demonstrate the engine capability to accept a reasonable amount of inlet flow distortion. The turbine engine is qualified in an altitude test cell under well controlled and accurately measured conditions. The aircraft inlet flow distortion is presently simulated by placing a distortion screen in front of the engine. A distortion screen consists of a heavy metal mesh of low blockage to which are attached one or more screens of higher blockage. The screen pattern is such that an engine inlet total pressure distortion pattern is produced which nearly duplicates a specified inlet distortion pattern. Unfortunately, because the pressure gradients produced by the distortion screen are a function of the airflow through the screen, the distortion screen will produce the desired distortion pattern at only one design airflow condition. Reference 5 is an excellent description of a good technique for the design of engine inlet distortion screens.

Using Figure 2 as an example, the distortion screen has an inner diameter of approximately 4 inches and an outer diameter of approximately 35 inches. The basic grid is spread across the

entire face of the screen and consists of 0.125 inch diameter wire in a square pattern of one wire per inch. The remaining screen mesh consists of 0.063 inch diameter wire in a square screen pattern with from two to nine wires per inch. These secondary screens are cut to approximate the isobars of the design distortion pattern and are not allowed to overlap each other. Although the secondary screens are safety-wired to the basic grid during screen development, they are brazed to one another and the basic grid after the design has been finalized.

Problem Definition

As would be expected, the flow downstream of the distortion screen is quite turbulent. The turbulence causes the pressure to oscillate about the average, or steady-state, pressure reading. The steady-state total pressure distortion will be less severe than the actual total pressure distortion that would appear transiently at the engine compressor face. Therefore, high response pressure instrumentation is required to accurately determine the flow distortion entering a turbine engine. The data obtained with the high-response pressure probes are referred to as dynamic data. However, high-response (approximately 800 Hz) pressure transducers, the associated hardware, and the increased data reduction requirements are very costly. Because of the increased costs of high response pressure distortion measurement, many Air Force systems are forced to use only steady-state pressure measurements to determine inlet screen distortion. A technique is needed to predict the worst-case dynamic distortion using

only steady-state pressure measurements. This paper summarizes an attempt to develop such a technique.

Reference 6 states that during the FB-111 flight test program, high pressure turbulence levels were associated with large gradients in the inlet total-pressure distortion pattern. This trend was judged to be indicative of regions that were reacting to large shear forces. It would therefore appear that a system for the measurement of the magnitude of pressure gradients would serve to approximate the amount of turbulence. If the amount of turbulence could be predicted, a factor might be devised which would serve to modify a distortion pattern measured with only steady-state pressure instrumentation. Should it succeed, then the steady-state pressure measurements alone would suffice to estimate the worst-case dynamic distortion pattern generated by a distortion screen.

Scope of Present Study

References 1 and 2 describe a system for calculating pressure gradients at the entrance to a turbine engine. A summation of these references is contained in a later chapter of this study. The pressure gradients are examined through the use of cylindrical coordinates. Equations are developed to calculate terms corresponding to the pressure gradients in the axial, radial and circumferential directions. The radial and circumferential pressure gradients are of primary importance to this study. Suffice it to say that the formulae for the radial and circumferential pressure gradients involve the axial velocity of the air entering

the compressor, the total pressure at the engine face, and the radial and circumferential pressure gradients at the compressor face. References 1 and 2 choose to call the resultant terms absolute vorticity. Hereafter the term vorticity will apply to the above definition. As the sum of radial and circumferential vorticity should be the best parameter to predict turbulence, the total vorticity was obtained by vector addition of the radial and circumferential vorticity.

High quality data were obtained from a distortion screen calibration effort and exposed to the hypothesis described above. Distortion was measured in terms of (maximum total pressure - minimum total pressure) / average total pressure at the engine face. This term is referred to as $\Delta p/p$. The dynamic pressure data were digitized so that the pressure readings from all 48 pressure probes were recorded simultaneously, with 0.00096 seconds between each data scan. Computer programs were used to scan 512 data scans to determine the worst-case pressure distortion levels for each data point. A computer program was devised to calculate and plot the total vorticity from the steady-state pressure measurements. The dynamic data were scanned to determine regions of the distortion pattern where the dynamic data varied most from the steady state values. For this study "turbulence" was defined as the standard deviation of the dynamic total pressure measurements from the steady-state total pressure measurements. An attempt was then made to correlate the "turbulence" levels with the difference between the steady-state and dynamic pressure

distortion patterns. A correlation was then attempted between the total vorticity and the "turbulence" levels. The final step was to attempt a direct correlation between the total vorticity and the difference between the dynamic and steady-state distortion patterns. The final objective was to develop an empirical technique to predict the worst-case total pressure distortion produced by an inlet distortion screen. The technique was to use only steady-state total pressure measurements and produce worst-case distortion predictions accurate enough to withstand the critical scrutiny of the U.S. Air Force and its engine manufacturers.

Chapter II Source of Data

Data Program

The data for this study were obtained from February 1 through April 6, 1973, by ARO, Inc., contract operator of the Arnold Engineering Development Center (AEDC), Arnold Air Force Station, Tennessee. The tests were performed under ARO project number RA 308 under Air Force Systems Command (AFSC) Element 64209F. (The tests were directed and funded by the F-15 Systems Program Office at Wright-Patterson Air Force Base, Ohio.) A copy of the letter authorizing the use of these data is contained in Appendix B.

Distortion Screens

The five distortion screens used in qualification testing of the F100-PW-100 turbofan engine were investigated. The data from the calibration of one distortion screen were discarded due to the low total pressures, and associated measurement inaccuracies, resulting from the simulated 58,600 ft. altitude / 0.9 Mach number (58.6K/0.9) flight condition of the test. Figures 2 through 5 are schematics which present the grid size and percent blockage of the remaining four screens. Each screen was required to produce a specified distortion factor at a corrected airflow, altitude and Mach number. Figure 1 is a description of the goal distortion patterns for the design of the four screens used in the test program. The objective of the screen design program was to duplicate the normalized pressure readings of these goal patterns to within 2% root-mean-squared error on a probe-by-probe basis. The

screens were found to conform to these criteria. Although data were generated at several airflow conditions for each of the four screens, the only data used for each screen were that generated at the design airflow.

Facility

The data were generated at the AEDC Engine Test Facility's J-1 test cell, which has an inner diameter of approximately 16 feet. The test hardware for this program consisted of the Pratt & Whitney Aircraft Company F-15 Aircraft Full-Scale Inlet Simulator and an engine simulator (cold pipe). A detailed schematic of the installation is shown in Figures 6 and 7. In Figure 6 air of a desired pressure and temperature entered the test cell from the left, as depicted by the arrow. The air first passed through a metal honeycomb and a screen which served as flow straighteners. Next the air passed through the venturies which served to measure the total cell airflow. All bypass and leakage air was subtracted from the total flow to obtain the airflow through the cold pipe. The flow exiting the venturies was broken up by air baffles before entering a low velocity plenum and conic screen. Next the air entered a transition duct to the F-15 inlet simulator. Air exited the inlet simulator through either the cold pipe engine simulator or the lower ramp bypass. Test cell pressure was maintained by facility exhausters which reduced cell pressure through the exhaust diffuser.

The F-15 inlet simulator, as sketched in Figure 7, was designed to simulate the flow at the engine face produced by the

actual F-15 aircraft inlet. The F-15 inlet uses three external ramps to create three oblique shocks prior to the normal shock during supersonic flight. The inlet simulator simulated only the end of the second ramp; the rest of the inlet to the cold pipe was an exact duplicate of the aircraft inlet. Through this configuration only the final oblique shock and the normal shock were simulated during supersonic flight. During simulation of actual inlet operation the lower ramp bypass and lower ramp position were regulated to obtain the proper Mach numbers and flow angles that would occur during flight. For this test program, the inlet simulator was configured to obtain the most uniform flow in the cold pipe inlet. A thorough description of the inlet simulator and its operation is contained in Reference 3.

The J-1 test cell, with the inlet simulator installed, was chosen for the screen calibration because dynamic data recording and processing systems were readily available to make the required time-dependent measurements and because no F-100 engine was available at that particular time period for testing in the J-1 test cell. For this test program all bleed and bypass lines were blanked off with the exception of the 36 inch lower ramp bypass line. Also, the fourth ramp bypass line was removed and the bypass valve closed, but with no blank off plate installed, and the primary air measuring venturis 3, 6, and 7 were blanked off. Unique test hardware for this program consisted of the following:

1. The environmental control system section used during the F100-PW-100 engine tests was replaced by a straight pipe

which connected the end of the subsonic duct to the instrumentation at the simulated engine face. A radial grid assembly was installed inside of this duct to which the distortion screens were attached. The screen face was 28.7 inches forward of the cold pipe flange.

2. To reduce the distortion of the flow field at the entry plane of the distortion screens, a flow straightening grid was installed approximately 58 inches upstream of the subsonic duct exit. Figure 8 is a schematic of the subsonic duct and shows the position of the flow straightening grid in the duct. This flow straightening grid was retained in the duct by bars which passed through the subsonic duct structure. Upon completion of the test program, the straightening grid and the support bars were removed and leak-tight plugs installed in the subsonic duct.
3. The cold pipe exit was fitted with a conic nozzle to effect airflow control. When required by test conditions, the nozzle area was changed by replacing the conic nozzle. Figure 9 is a schematic of the conic nozzle installation and also documents the areas and the area ratios of the conic nozzles used during this test program.

Test Conditions

The test conditions for each screen are as described in Table 1.

Test / Data Point	Screen	Alt./Mach	P ₂ psia	T ₂ °R	P _{s2} psia	Corrected Airflow
HA-08/10	4-5F	65.5K/2.2	6.90	769	6.21	147.1 (lb/sec)
HA-16/10	5-1F	68K/2.5	9.96	800	9.10	128.1
HA-17/12	1-7F	30K/0.9	6.61	481	4.77	229.3
HA-18/24	3-5F	64K/1.6	3.30	590	2.72	200.7

Table 1

P₂ is defined as the average total pressure at the engine inlet, P_{s2} is the average static pressure at the engine inlet, T₂ is the total temperature of the air entering the engine, and the corrected airflow refers to the (mass flow entering the engine) x $\sqrt{T_2/519}$ / (P₂/14.7).

Instrumentation

The engine/inlet interface plane was defined with a total pressure and total temperature array as shown in Figure 10. Each total pressure location involved both a high response (up to 800 Hz) Kulite transducer and a close-coupled, slow response (40 Hz) Statham transducer. For this test Kulite model XCEL-1 1/4 inch transducers were used. These transducers had a 0-50 psia range and had a small fine-meshed screen incorporated into each transducer mount to block any rust particles picked up in the inlet air heaters. The Statham transducers were model PA8569, with a 0-50 psia range. The Statham transducers were mounted in the bulletnose of the simulated engine and temperature conditioned with nitrogen to 200 ± 10°F. The Kulite transducers were exposed directly to the freestream total pressure and were not temperature conditioned.

Signal Conditioning

An electronic system was developed by Pratt & Whitney Aircraft to compensate for the intercept shift of the Kulite volt/psia calibration curve. It was assumed that the slope of the calibration curve would remain constant within the temperature envelope of this program. Figure 11 is a diagram of this electronic system. It was used to merge the high response Kulite signal with the low response Statham signal. The Statham signal was passed through a 1/4 Hz low pass filter and the Kulite signal was passed through a 1/4 Hz high pass filter. The resultant signals were merged and normalized to form a pressure signal independent of the Kulite shift in mean pressure with temperature. The merge equipment outlined in Figure 11 was enclosed in a temperature conditioned room. The final pressure signal was passed through a low pass filter which was designed to the filter characteristics described in Figure 12. This curve was developed by Pratt & Whitney Aircraft to account for the dwell time required for a compressor blade to respond to inlet distortion. The analog signal was finally digitized at one cut every 0.00096 seconds. For each data point analyzed for this study, 512 time slices were scanned.

Chapter III Review of Vorticity Theory

Nomenclature

Characters:

a	speed of sound
D	engine inlet diameter
e	unit directional vector
H	enthalpy
p	pressure
R	gas constant (gravity included)
s	specific entropy
S	entropy
t	time
T	temperature
x	vector cross product
∂	partial derivative
∇	vector operator del
γ	ratio of heat capacities
ω	vorticity

Subscripts:

1	station 1
2	station 2
r	radial direction
z	axial direction
θ	circumferential direction

Discussion

The earliest reference promulgating the use of relative vorticity for the assessment of turbine engine stability is Reference 1. A summation of Reference 1 is contained in Reference 2. As an understanding of the basic tenets of the proposed vorticity approach is essential for this study, this chapter will briefly summarize References 1 and 2. The theory and equations in this chapter do not represent original work and are obtained from these references. A computer program to perform the vorticity calculations was developed for this study and does repre-

sent an original effort, although some of the subprograms have been adapted from different programs not originally written by the author.

In general form, Crocco's Theorem can be written as:

$$TVS + ux(\nabla xu) = \nabla H + \partial u / \partial t$$

From basic thermodynamics:

$$\frac{s_2 - s_1}{R} = -\ln\left(\frac{p_{t2}}{p_{t1}}\right) + \frac{\gamma}{\gamma-1} \ln\left(\frac{T_{t2}}{T_{t1}}\right)$$

For a constant total enthalpy, the combination of the above two equations yields:

$$-\frac{RT}{p_t} \nabla p_t + ux(\nabla xu) = \frac{\partial u}{\partial t}$$

The above equation can be rewritten in cylindrical coordinates.

The component for the r direction is:

$$-\frac{RT}{p_t} \frac{\partial p_t}{\partial r} + (u_\theta \omega_z - u_z \omega_\theta) = \frac{\partial u_r}{\partial t}$$

and for the θ direction is:

$$-\frac{RT}{p_t} \frac{1}{r} \frac{\partial p_t}{\partial r} + (u_z \omega_r - u_r \omega_z) = \frac{\partial u_\theta}{\partial t}$$

Let $p_t = p' / \bar{p}_t$, $u = u' \bar{a}$, $r = r' D / 2$ and $\omega = \omega' 2 \bar{a} / D$, where \bar{p}_t is the average total pressure at the engine face, \bar{a} is the average speed of sound at the engine face, D is the compressor diameter, and ω is the fluid vorticity. Rewriting the equations for radial and circumferential vorticity yields:

$$-\frac{1}{p'} \frac{\partial p'}{\partial r'} + \frac{\bar{a}^2}{a^2} (u_\theta' \omega_z' - u_z' \omega_\theta') = \frac{V_{tip} \bar{Na}}{2\pi a^2} \frac{\partial u_r'}{\partial t'}$$

and

$$-\frac{1}{\gamma p'} \left(\frac{1}{r'} \frac{\partial p'}{\partial \theta} \right) + \frac{\bar{a}^2}{a^2} (u_z \omega_r - u_r \omega_z) = \frac{V_{tip} \bar{Na}}{2\pi a^2} \left(\frac{\partial u_\theta'}{\partial t'} \right)$$

where N is the number of blades, V_{tip} is the velocity of the compressor blade tip, and $t = t' \pi D / (V_{tip} N)$ and is a function of the blade passing frequency.

After considerable order-of-magnitude analysis, the above equations reduce to:

$$\omega_r' = \frac{1}{\gamma p' u_z'} \left(\frac{1}{r'} \frac{\partial p'}{\partial \theta} \right)$$

and

$$\omega_\theta' = \frac{1}{\gamma p' u_z'} \left(\frac{\partial p'}{\partial r'} \right)$$

but $u_z' = u_z/a = u_z/(\bar{u}_z/\bar{M}) = \bar{M}u_z/\bar{u}_z =$

$$\bar{M} \left(\frac{(p_t - p_s)(2/\rho)}{(\bar{p}_t - \bar{p}_s)(2/\bar{\rho})} \right)^{.5} = \bar{M} \left(\frac{p_t - p_s}{\bar{p}_t - \bar{p}_s} \right)^{.5}$$

Recalling that $\omega_r' = \omega_r D/2a$ and $\omega' = \omega_\theta D/2a$, the final equations are obtained:

$$\omega_r = \frac{2a}{\gamma D p' \bar{M} Q} \left(\frac{1}{r'} \frac{\partial p'}{\partial \theta} \right) \quad \frac{\text{ft.}/\text{sec.}}{\text{ft.}}$$

and

$$\omega_\theta = \frac{2a}{\gamma D p' \bar{M} Q} \left(\frac{\partial p'}{\partial r'} \right) \quad \frac{\text{ft.}/\text{sec.}}{\text{ft.}}$$

where

$$Q = \left(\frac{p_t - p_s}{\bar{p}_t - \bar{p}_s} \right)^{.5}$$

The final equations described above are those which were programed into the automated calculation procedure. A listing of the computer program is contained in Appendix A.

As each of the terms for radial and circumferential vorticity are vectors, the absolute value of the total vorticity was obtained through the Pythagorean theorem. The absolute value of the total vorticity, hereafter termed the normal vorticity, was deemed of primary importance as this value should correlate directly to the total pressure gradient at the engine face. As stated previously, the total pressure gradient should correlate with the turbulence at the engine face.

Chapter IV Data Analysis

Steady State Data

As described in Chapter II, the data from the calibration of four distortion screens were evaluated. Only the data generated at the screen design airflows were used. The steady state data were processed through the automated calculation procedure to obtain values of normal vorticity.

Transient Data

Next the high-response total pressure data, termed dynamic data, were processed. As described in Chapter II, each dynamic data point consisted of 512 scans, digitized at 0.00096 seconds/cut, which had been processed and filtered according to Figure 12. Each dynamic data point was scanned to determine the worst case of distortion. As described in Chapter I, the criterion for the worst distortion pattern was (maximum total pressure - minimum total pressure)/average total pressure at the engine face, commonly referred to as $\Delta p/p$. The distortion patterns for the worst cases were obtained through use of the computer program listed in Appendix A. Bad Kulite readings were determined by visual inspection of the raw data. The computer program substituted values for the bad Kulite readings by using a weighted average of the surrounding pressure probes.

Turbulence

Next, the standard deviation of each high response pressure probe from the steady-state pressure reading was obtained. The standard deviation of the high response pressure data was termed

"turbulence." "Turbulence" patterns and average "turbulence" values were obtained.

Vorticity

Finally, the steady-state pressure data were processed through a computer program developed to calculate and plot the normal vorticity at the engine inlet. A listing of this program is contained in Appendix A. The computer print-out from the four input cases immediately follows the listing. The development of the principal vorticity equations is contained in Chapter III.

Analysis of Data

Figures 13 through 16 compare the steady-state pressure distortion pattern, the worst-case pressure distortion pattern, the pressure "turbulence" distortion pattern, and the normal vorticity pattern for each of the four screens. A comparison of the steady-state and the worst-case pressure distortion patterns shows that the isobar of average pressure changes little. Also, the average compressor face total pressure does not change between the steady-state and worst-case pressure distortion. Therefore, all that need be done is to determine a modifier to increase the above average steady-state pressure readings and decrease the below average pressure readings, while maintaining the same average pressure. If such a modifier could be obtained, an accurate estimate of the worst case distortion pattern would be easily and economically available.

Next, the standard deviation of the high response pressure from the corresponding steady state pressure, termed "turbulence,"

and the normal vorticity patterns were compared. The regions of high and low pressure "turbulence" do not compare well with the regions of high and low normal vorticity. Therefore, normal vorticity alone is not an accurate predictor of "turbulence" level. Also, there is no evidence to indicate that average normal vorticity correlated with average "turbulence" level.

The lack of correlation between normal vorticity and "turbulence" is not understood. Some high frequency "turbulence" could result from the airflow through the distortion screen. This type of "turbulence" would not be related to normal vorticity. The distortion screen mesh size is assumed to have little effect on the "turbulence" level however, since the 28.7 inches between the distortion screen and the pressure instrumentation would permit considerable dissipation of the fine grained "turbulence." Any remaining fine grained "turbulence" would be eliminated in filtering the data according to Figure 12.

In an attempt to correlate the average "turbulence" level with average normal vorticity and the difference between steady state and dynamic $\Delta p/p$, Table 2 was developed.

Test / Data Point	Average Turbulence	$\Delta p/p$ Difference	Ave. Normal Vorticity	Steady State $\Delta p/p$
HA-08/10	0.042	0.0266	652	0.2213
HA-16/10	0.092	0.0523	610	0.1774
HA-17/12	0.070	0.0283	383	0.2627
HA-18/24	0.028	0.0375	484	0.1766

Table 2

Examination of Table 2 reveals that nothing correlates on macroscopic terms. The objective of this study was to have the

column of the difference between steady state and dynamic $\Delta p/p$ correlate with the average normal vorticity and the average "turbulence." As described above, average normal vorticity is not a good indicator of average "turbulence." Table 2 demonstrates that average normal vorticity is not a good correlator of $\Delta p/p$ difference either.

Analysis of Probe-by-Probe Data

Next, the correlation between $\Delta p/p$ difference and normal vorticity was approached on a probe-by-probe basis. The pressures in the distortion patterns were normalized by dividing them with the average total pressure of the entire distortion pattern. The difference between the lowest normalized pressure from the steady-state pressure distortion pattern and the lowest normalized pressure from the worst-case pressure distortion pattern was obtained. This difference was compared to the normal vorticity at the location of the lowest pressure reading from the steady-state distortion map. In a similar manner, normal vorticity was compared with the difference between the highest normalized pressure readings from the steady-state and worst-case pressure distortion patterns. Figure 17 summarizes the results of this comparison.

Although the normal vorticity values do appear to be a function of the normalized pressure difference, the correlation is not good enough to predict the worst-case distortion to within 1% $\Delta p/p$. Of particular concern is the negative slope of the data points from test HA-18. The characteristic of the HA-18 data prohibits further analysis of the available data.

Chapter V Recommendations for Additional Analysis

Shortcomings of Present Analysis

Data storage problems have prohibited additional data analysis. Flaws in the problem approach and data analysis are presented to enhance additional studies in this field and to indicate weak areas of this study which may have led to unjustified conclusions.

The objective of this study was to develop a technique for predicting the worst-case pressure distortion produced by a distortion screen at the entrance of a gas turbine engine. The criterion used to determine the worst-case pressure distortion was found to be flawed and should be changed in any additional studies. Distortion severity is measured by the depth, extent, and location of the pressure defect at the engine face. For this study, however, engine characteristics were ignored so that distortion location is not of importance. The criterion used to determine worst-case pressure distortion was $(\text{maximum total pressure} - \text{minimum total pressure}) / \text{average total pressure at the engine face}$. This criterion is a poor measure of the depth and extent of pressure distortion and should be improved in additional efforts. A step in the right direction would be the use of $(\text{average total pressure} - \text{minimum total pressure}) / \text{average total pressure at the engine face}$. Although this criterion will improve the evaluation of the depth of the pressure defect, it does not address the extent of the pressure defect and could be improved further.

Insufficient statistical tools were exercised during the evaluation of the probe-by-probe data. For this study the difference between the lowest normalized pressure from the steady-state pressure distortion pattern and the lowest normalized pressure from the worst-case pressure distortion pattern was obtained. This difference was compared to the normal vorticity at the location of the lowest pressure reading from the steady-state distortion pattern. This approach was too simplified for a problem as statistical as screen-induced pressure distortion. In some instances, when the location and value of the lowest normalized pressure on the steady-state distortion pattern had been determined, an examination of the worst-case distortion pattern revealed that the normalized pressure at the same location had actually increased, although the total distortion level had worsened. Any additional studies should include an improved approach to comparing a worsening of pressure distortion levels with normal vorticity.

This study has emphasized the vorticity at the locations of the lowest and highest total pressure in the steady-state distortion pattern. If the amount of "turbulence" in screen-induced distortion occurs in the regions of the highest pressure gradient, the lowest normalized pressure in the worst-case distortion pattern may well occur at a point different than the point of lowest normalized steady-state pressure. Therefore, to obtain a more accurate estimate of the worst-case pressure distortion using only steady-state pressure measurements, the vorticity at each

location should be applied in some fashion to the steady-state distortion map.

Improved Test Program

The objectives of the screen calibration program were substantially different than those of this study. To obtain a more worthwhile effort, a test program should be developed and directed towards the prediction of worst-case screen-induced pressure distortion. The improved test program would produce data which would enhance a distortion prediction effort.

First, several screens should be tested. The screens should cover a large range of depth, extent and location. The distortion screens should consist of an array of classical distortion patterns, such as 180° circumferential distortion and radial distortion located at both the hub and tip of the imaginary fan blades, as well as complex distortion patterns resembling those generated by an actual aircraft inlet. Data should be obtained from the distortion screens over a range of Reynolds Number Index and corrected airflow (as defined in Chapter II). The breadth of this proposed test plan should provide the data required for the most rigorous analysis.

Second, every effort should be made to assure correct data. Pressure probes should not be located behind seams in the distortion screen where weld beading could produce non-representative "turbulence." The signals from the high-response pressure transducers should be carefully analyzed to verify that electronic noise is not degrading the signal quality. In short, every effort

should be made to assure the highest quality data.

Third, all data should be generated with a "cold-pipe" engine rather than an actual gas turbine engine. The advantages of a "cold-pipe" engine are twofold. First, there would be no need to expose an engine to the high levels of distortion that would be covered in a complex test matrix. The level of risk, as well as data reduction requirements and test cost, could therefore be reduced. Second, use of a "cold-pipe" engine would ensure a uniform static pressure at the distortion measurement plane. Data generated under uniform pressure conditions would apply to several different engines, where data generated behind a certain engine and its peculiar flow redistributions may apply only to one engine or family of engines.

Chapter VI Conclusions

The objective of this study was to develop an empirical technique using normal vorticity to predict the worst-case total pressure distortion produced by an inlet distortion screen at the entrance of a gas turbine engine. That objective has not been met. Modifications to the local computer system have caused the data tapes to be unreadable and additional study of the data is not possible. Nevertheless, several conclusions can be drawn from the studies.

Bulk quantities, such as average normal vorticity or average turbulence, do not relate to the differences between the steady-state and worst-case pressure distortion patterns. These differences must be estimated using probe-by-probe evaluation of the steady-state data.

The available data do not indicate any correlation between normal vorticity and the standard deviation of the high-response pressure measurements. The lack of correlation may be caused by phenomena not yet understood. Other causes may be that high-response pressure probes were located behind the weld at the screen interfaces and experienced non-representative "turbulence," or that electromagnetic interference or faulty pressure probes compromised the quality of the data. Future studies should choose probe locations carefully and scrutinize the data so that areas of examination can be narrowed to physical phenomena.

Limited data has indicated that there is a relationship between normal vorticity and the differences between the steady-

state and worst-case pressure distortion patterns. This relation appears to be influenced by Reynolds Number Index (RNI). (See Figure 6.) Further study should be performed to confirm or refute this relationship.

Finally, this study has outlined a test program which would complement an effort to predict worst-case pressure distortion from only steady-state pressure measurements. Normal vorticity has been shown to be related to the difference between steady-state and worst-case screen-induced distortion. Efforts to refine this relationship should prove successful with improved data and additional studies.



Figures



Goal Distortion Patterns for Screen Design

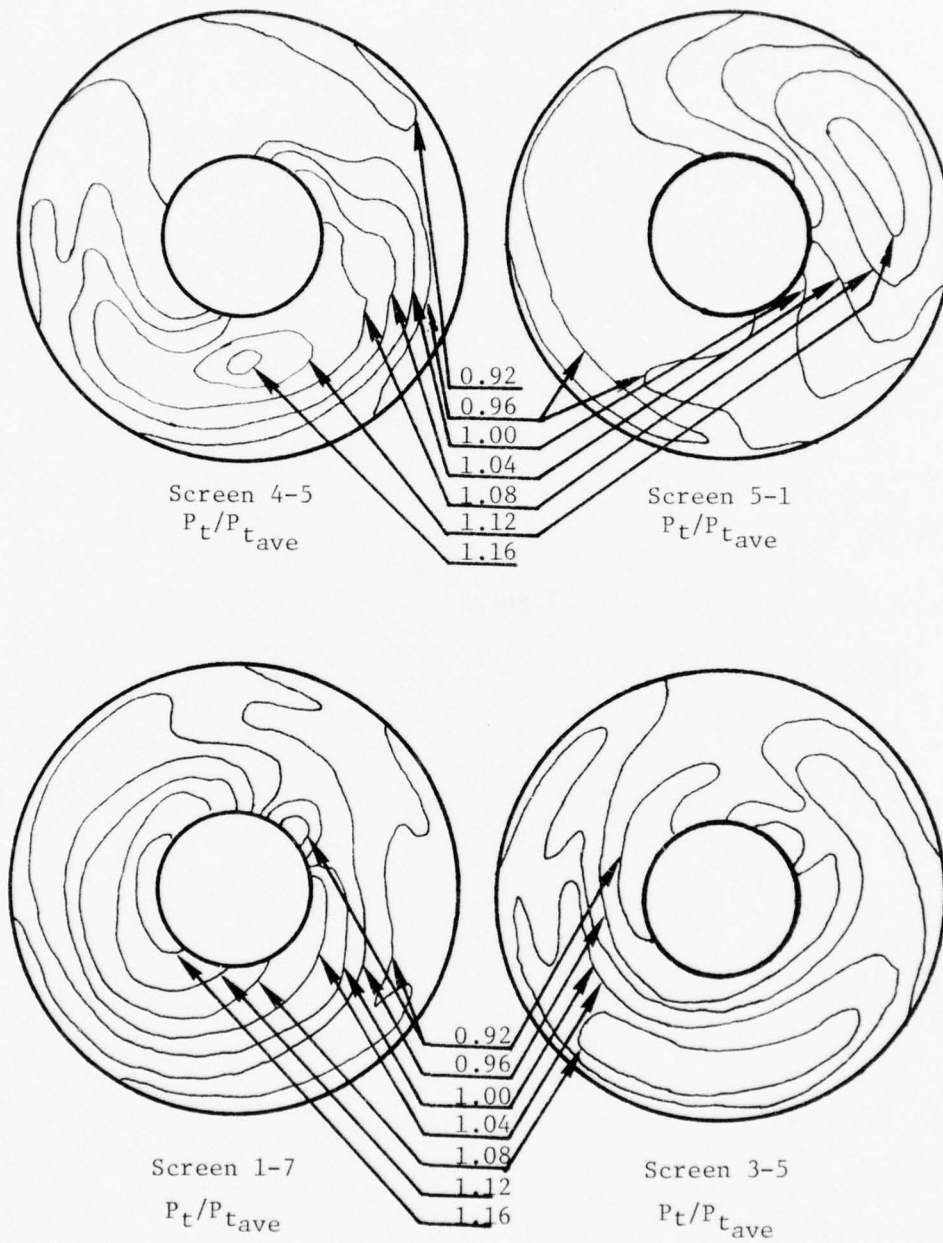
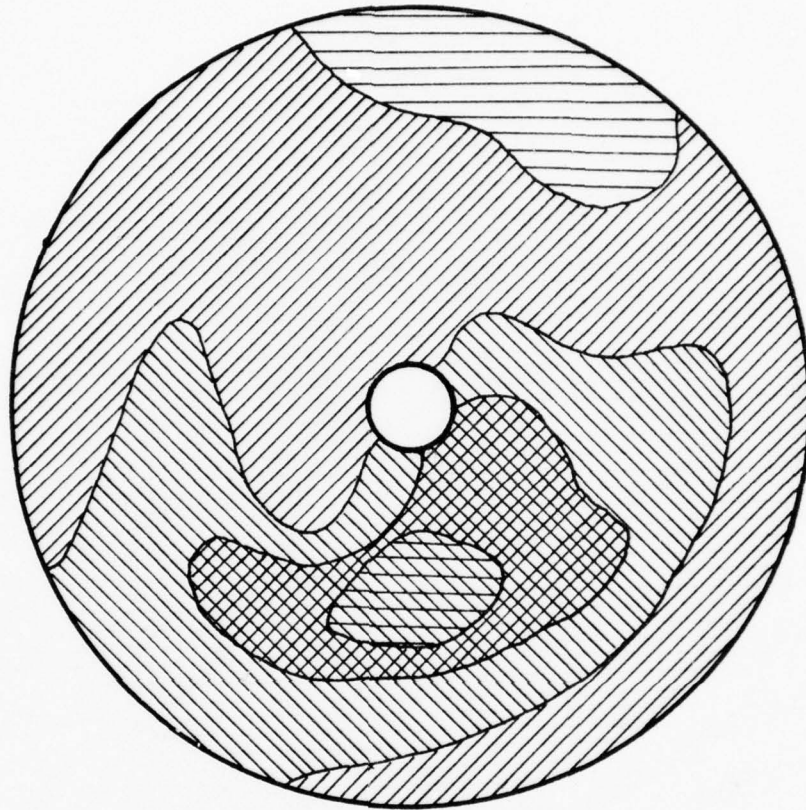


Figure 1

Screen Blockage and Grid Size

Screen 4-5F








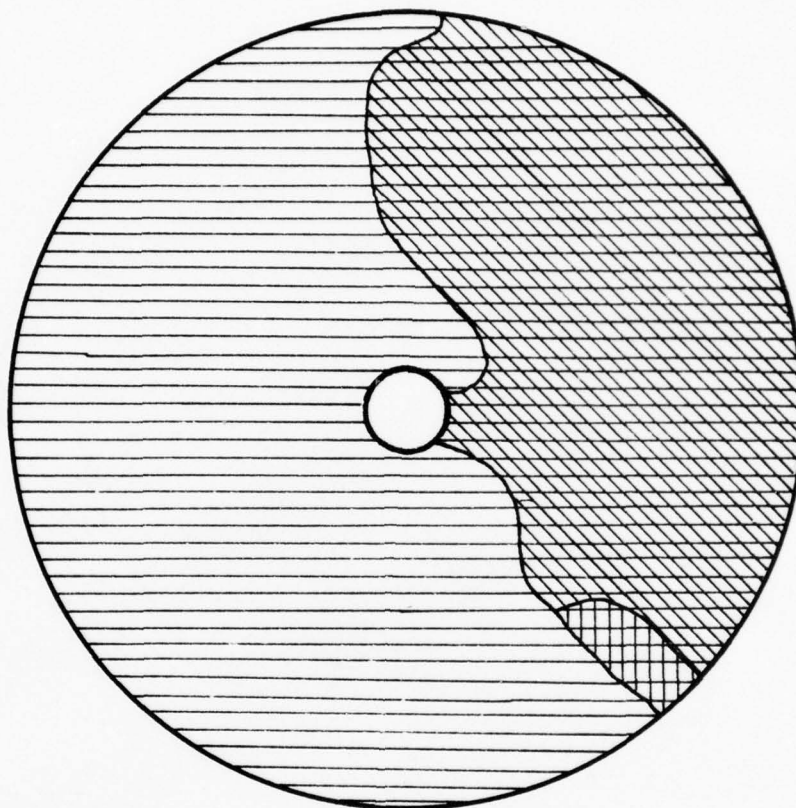
	% Total Area	% Blockage	Screen Size
	3.5	23.9	1 x 1 x .125 (Basic Grid)
	13.6	23.6	2 x 2 x .063 inches
	24.8	34.4	3 x 3 x .063 inches
	49.5	78.1	8½ x 8½ x .063 inches
	8.7	81.3	9 x 9 x .063 inches

Figure 2

Screen Blockage Area and Grid Size

Screen 5-1F

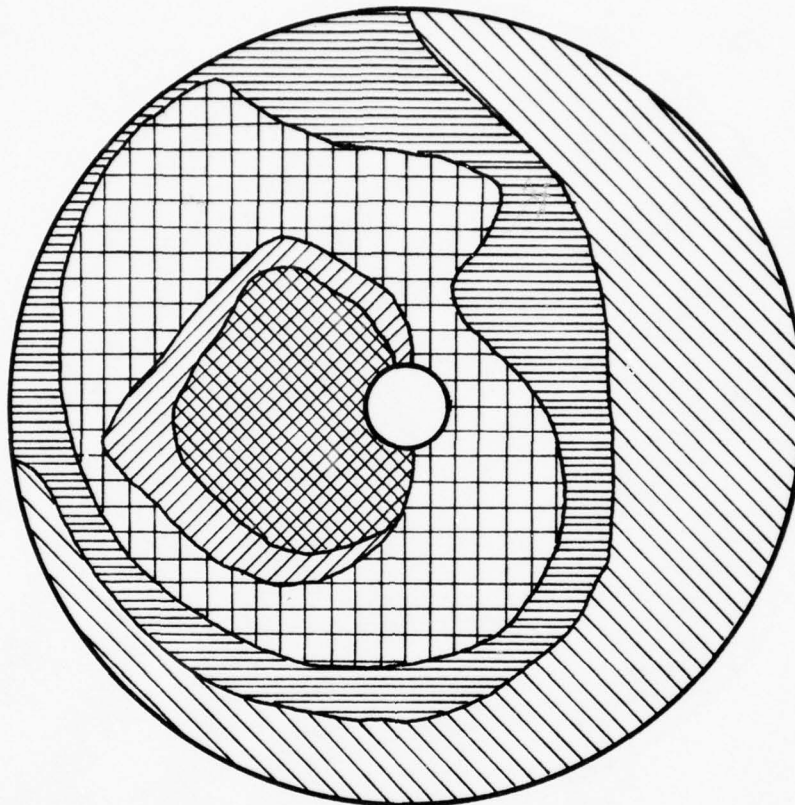


% Blockage		Screen Size	
	53.1	5 x 5 x .063 inches	
	86.3	10 x 10 x .063 inches	
	23.9	1 x 1 x .125 (Basic Grid)	

Figure 3

Screen Blockage and Grid Size

Screen 1-7F








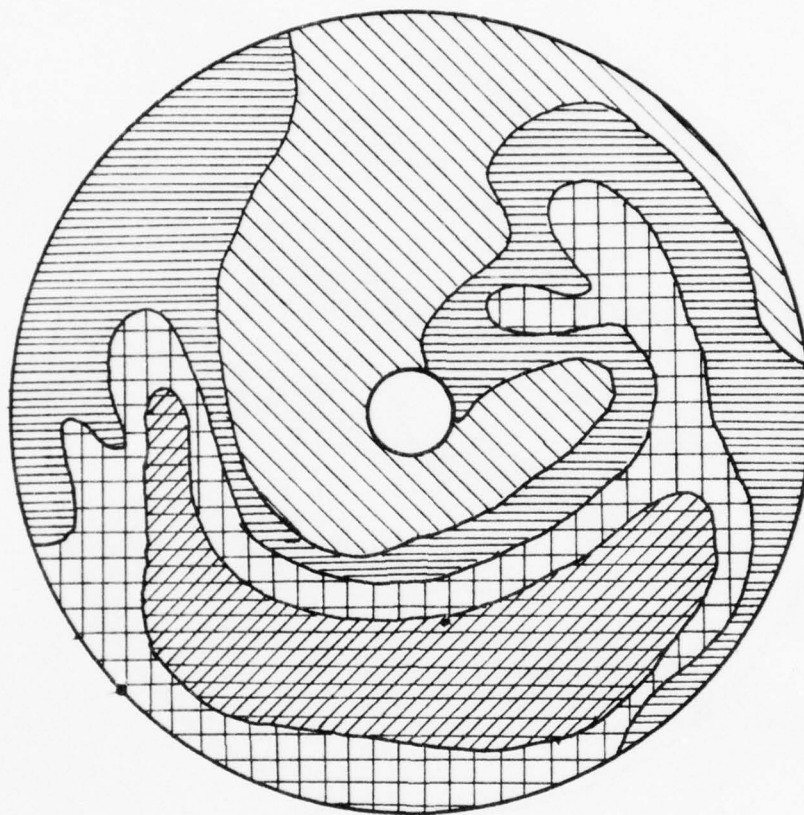
	% Total Area	% Blockage	Screen Size
	9.2	23.9	1 x 1 x .125 (Basic Grid)
	6.4	23.6	2 x 2 x .063 inches
	29.3	29.0	2½ x 2½ x .063 inches
	20.6	53.1	5 x 5 x .063 inches
	34.5	57.2	5½ x 5½ x .063 inches

Figure 4

Screen Blockage and Grid Size

Screen 3-5F




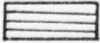


% Total Area		% Blockage	Screen Size
	31	57.2	$5\frac{1}{2} \times 5\frac{1}{2} \times .063$ inches
	25	48.8	$4\frac{1}{2} \times 4\frac{1}{2} \times .063$ inches
	23	29.0	$2\frac{1}{2} \times 2\frac{1}{2} \times .063$ inches
	21	23.9	$1 \times 1 \times .125$ (Basic Grid)

Figure 5

Test Cell, Schematic of Installation

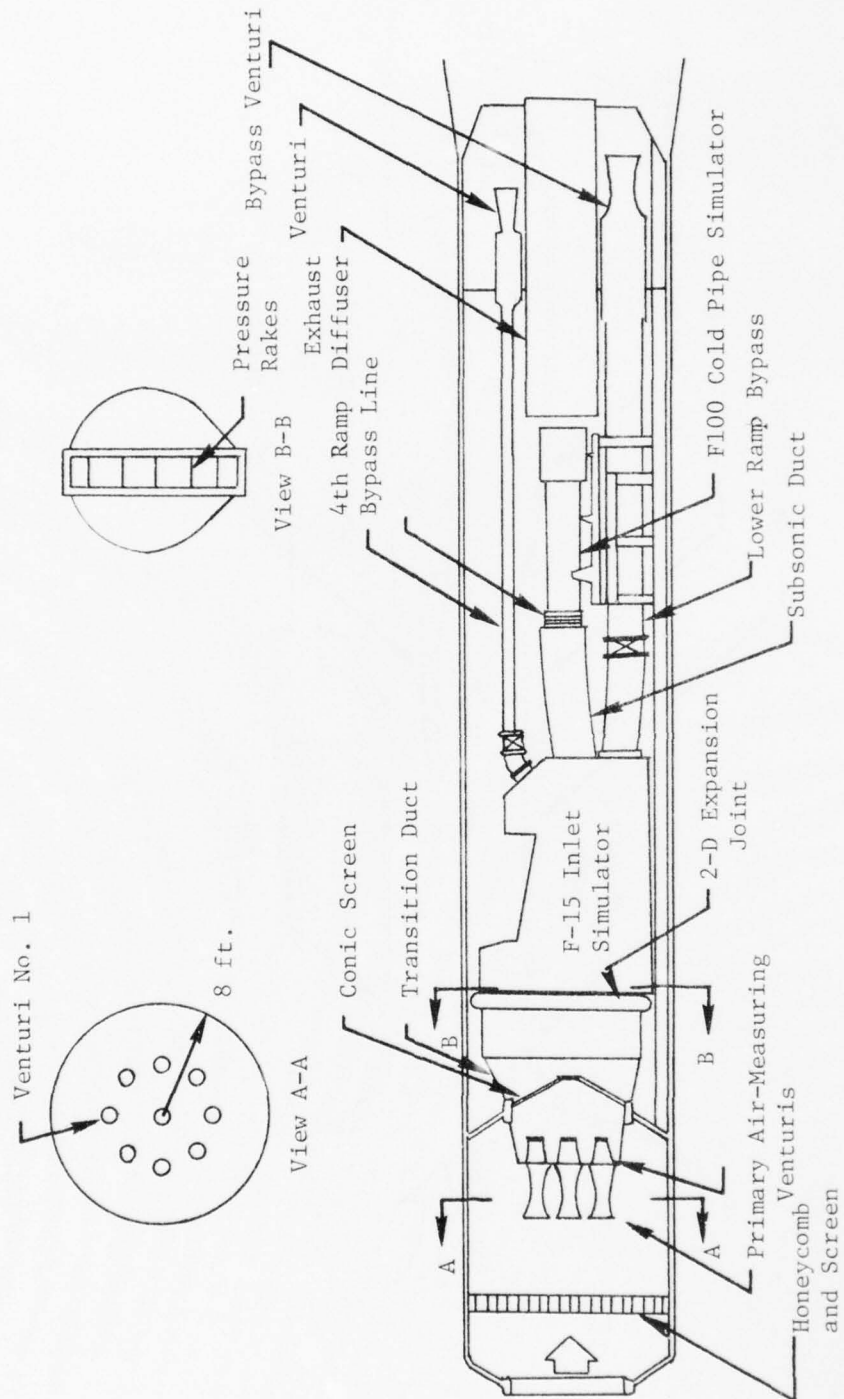
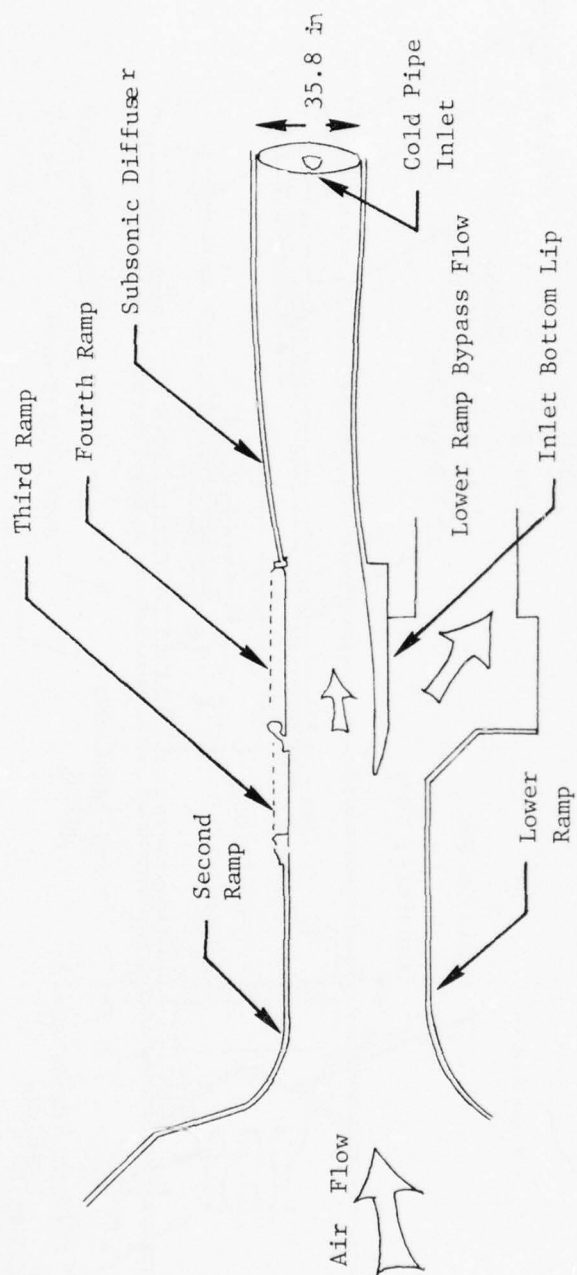
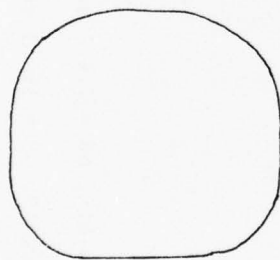
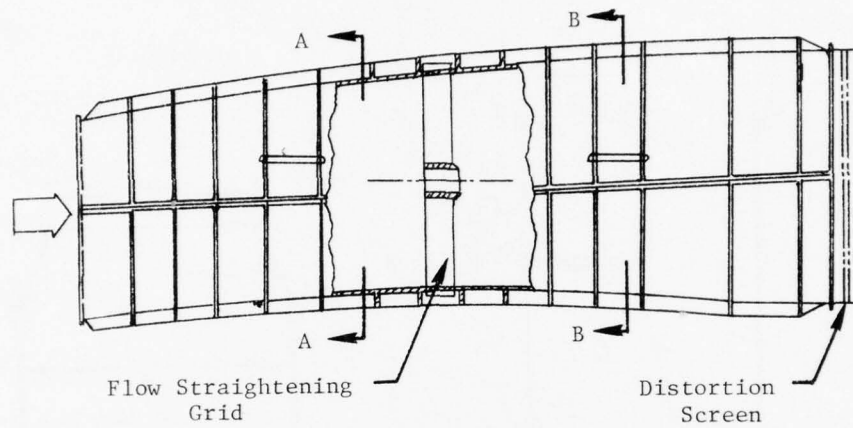


Figure 6

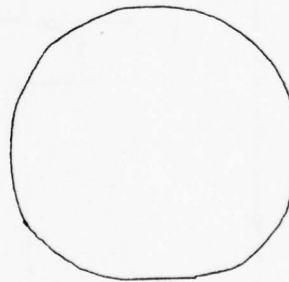
Schematic of Inlet Simulator



Schematic of Subsonic Duct



Section A - A



Section B-B

Figure 8

Schematic of the Cold Pipe

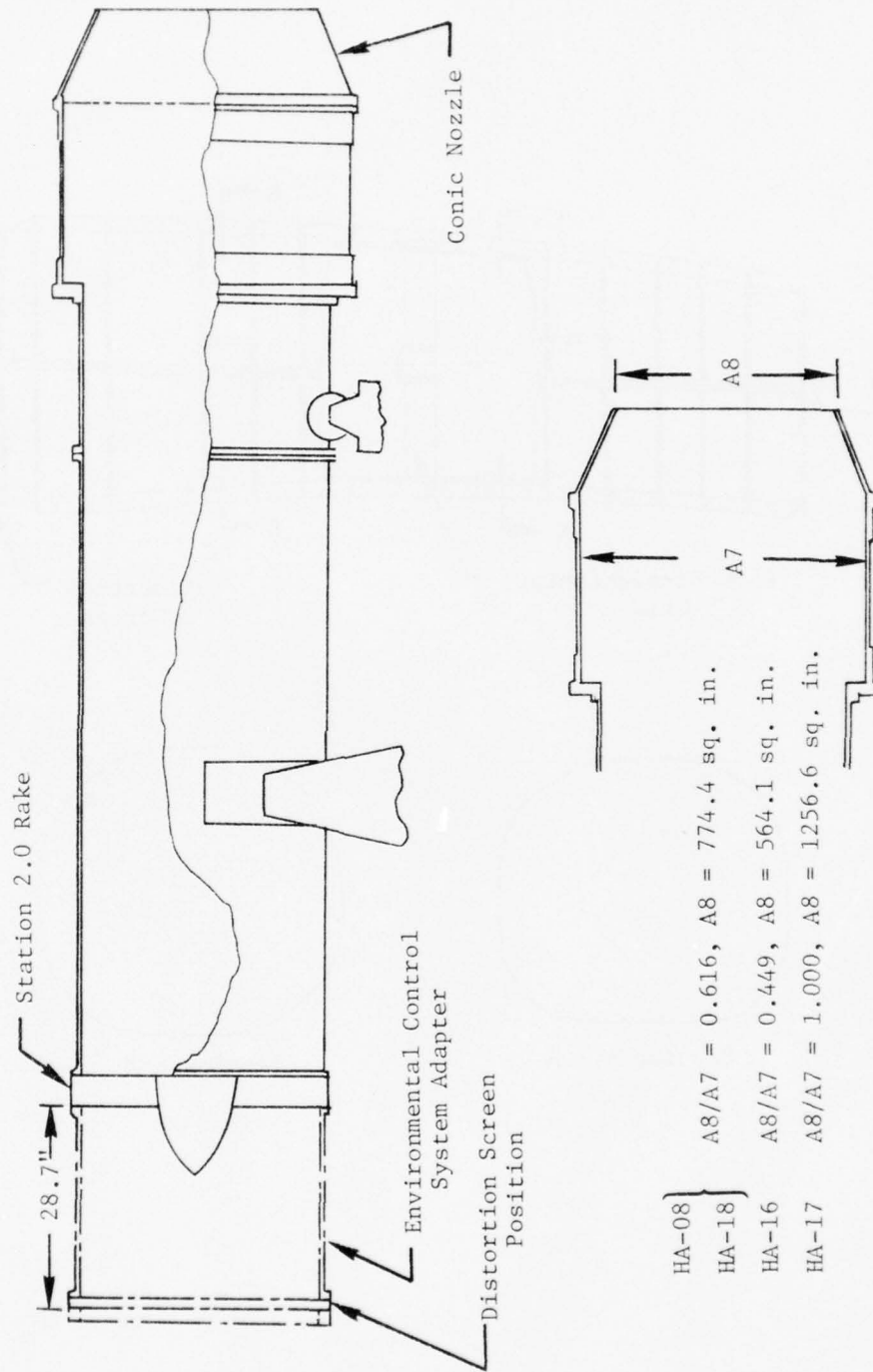
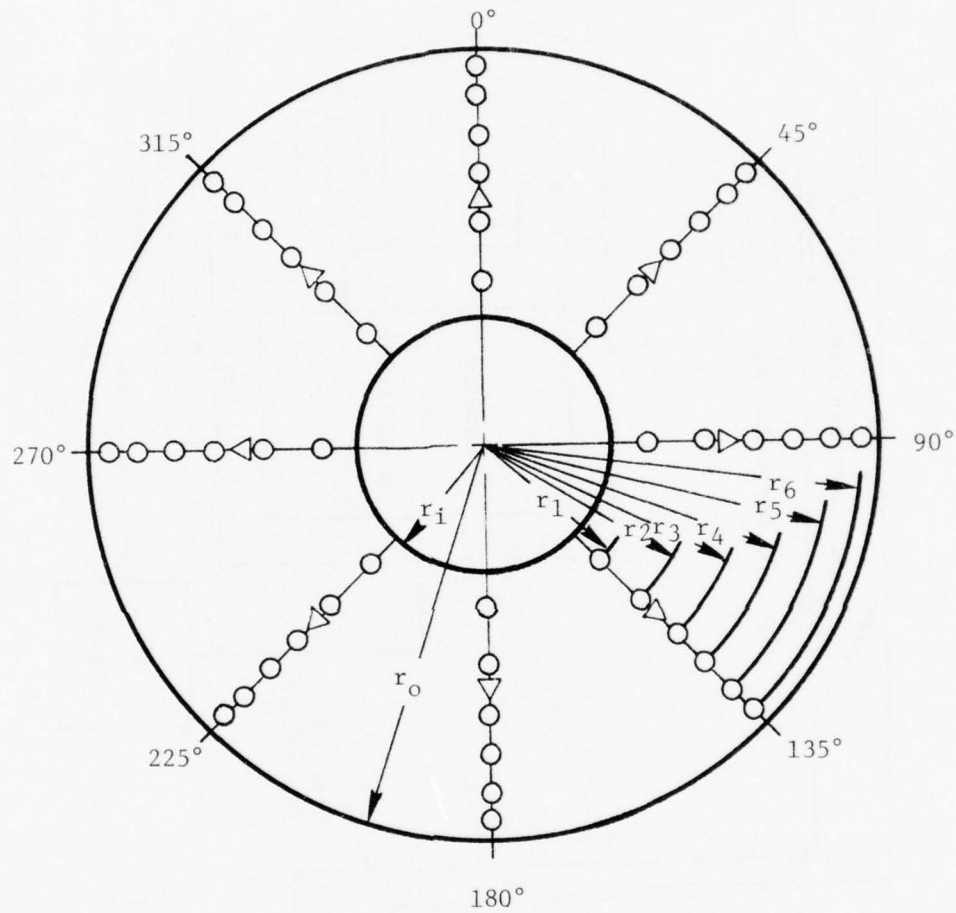


Figure 9

Engine Inlet Instrumentation Diagram

(View Looking Upstream)



- Total Pressure
- △ Total Temperature

r_i = 5.65 inches
 r_1 = 7.22 inches
 r_2 = 9.92 inches
 r_3 = 11.99 inches
 r_4 = 13.76 inches
 r_5 = 15.32 inches
 r_6 = 16.74 inches
 r_o = 17.39 inches

Figure 10

Data Conditioning System

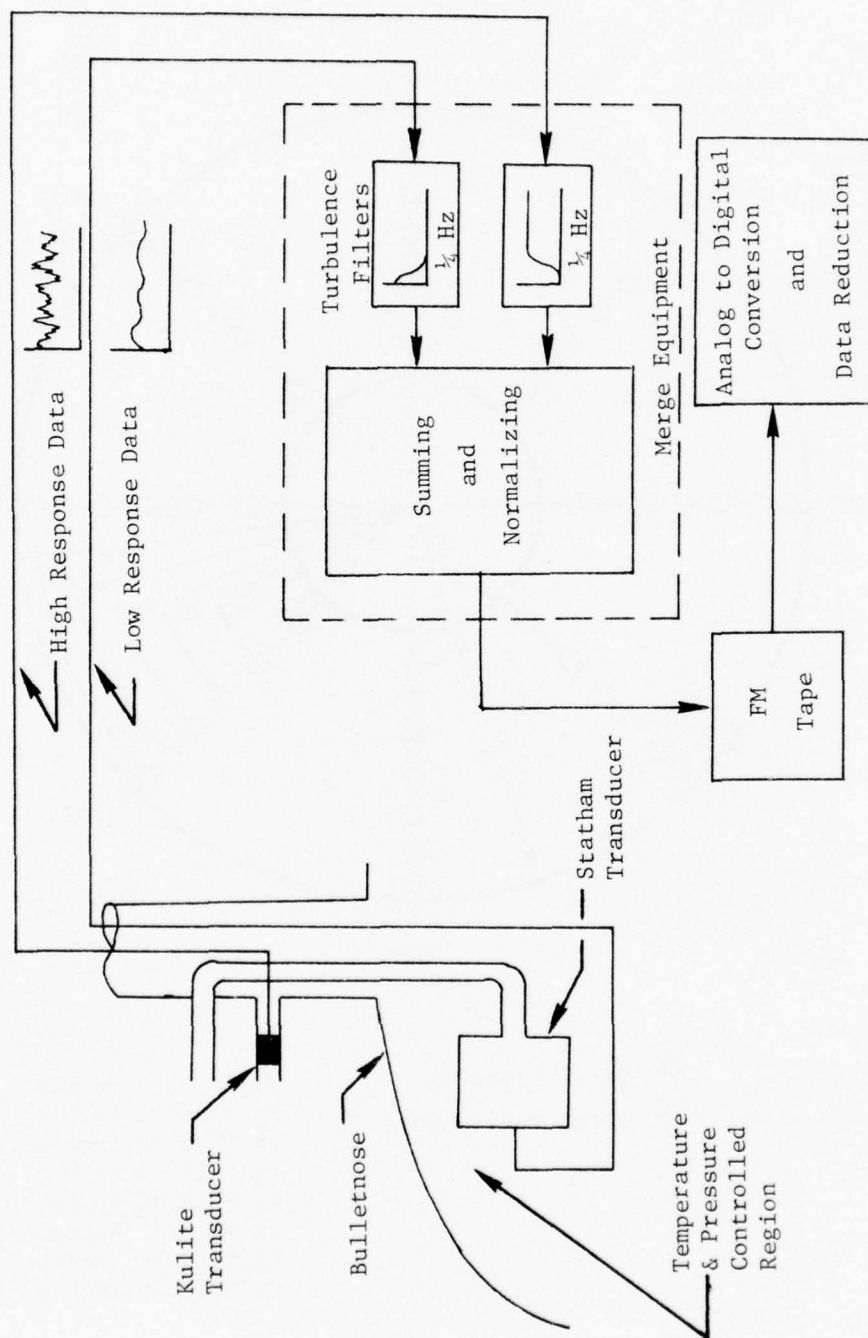


Figure 11

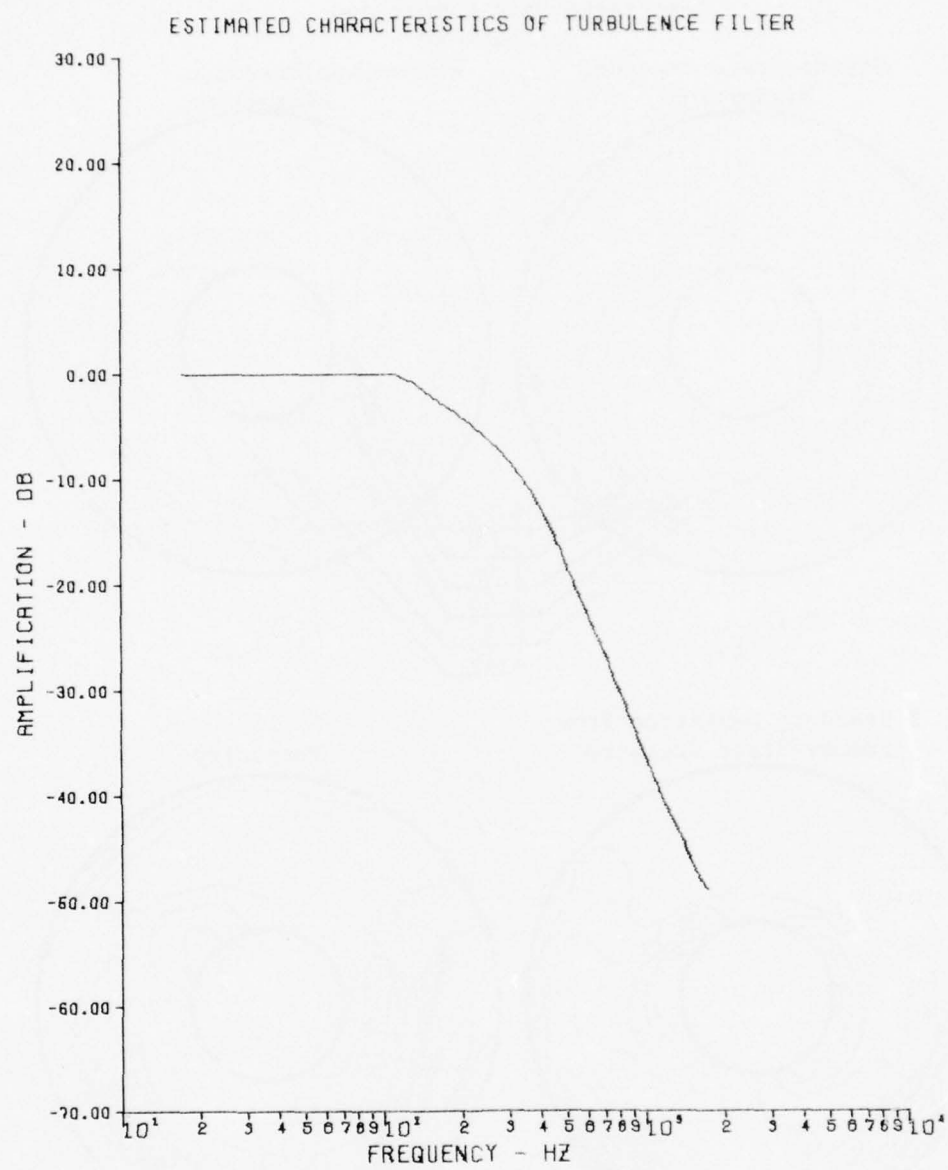


Figure 12

Test HA-08 Data Comparison
Screen 4-5F

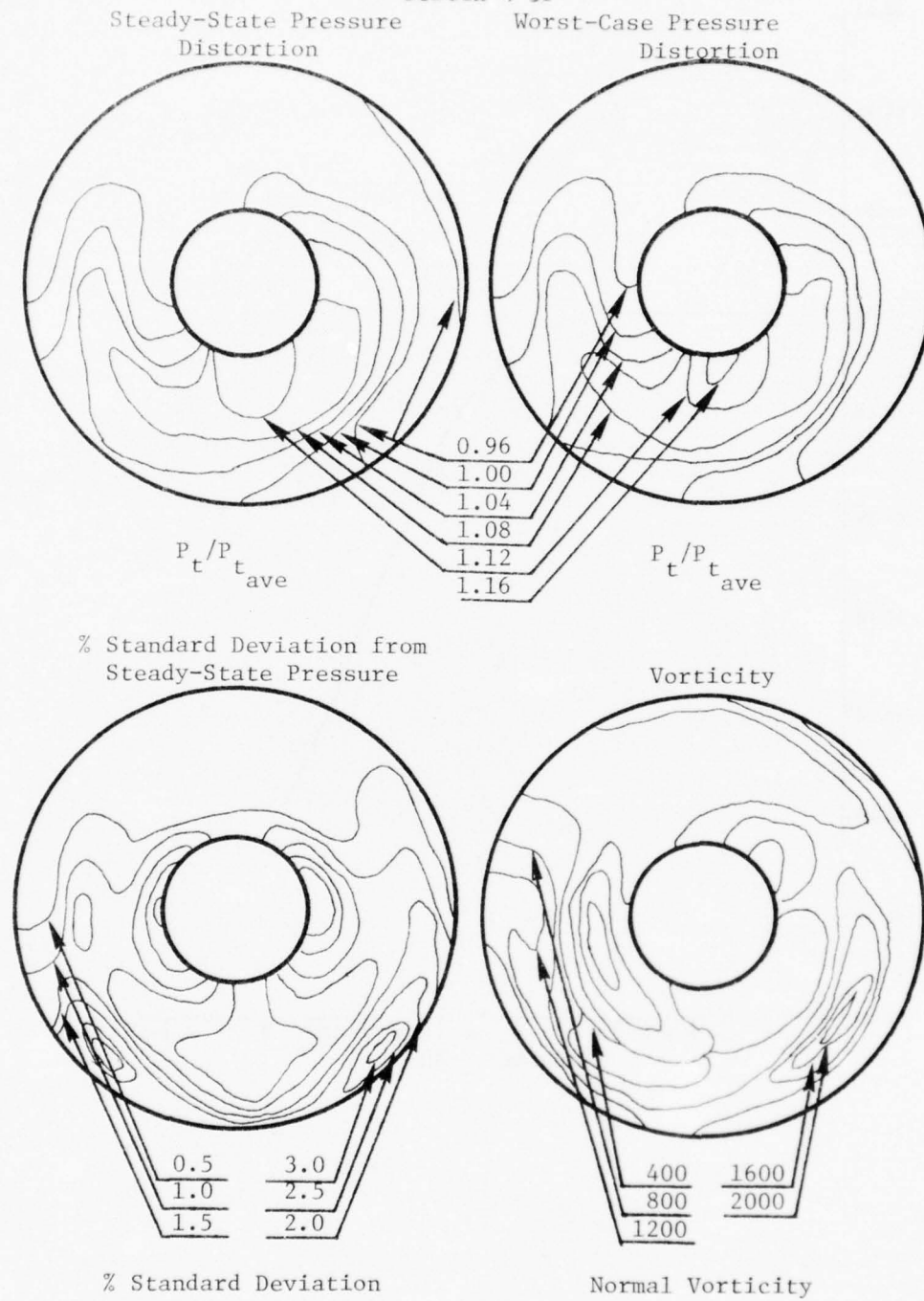


Figure 13

Test HA-16/10 Data Comparison
Screen 5-1F

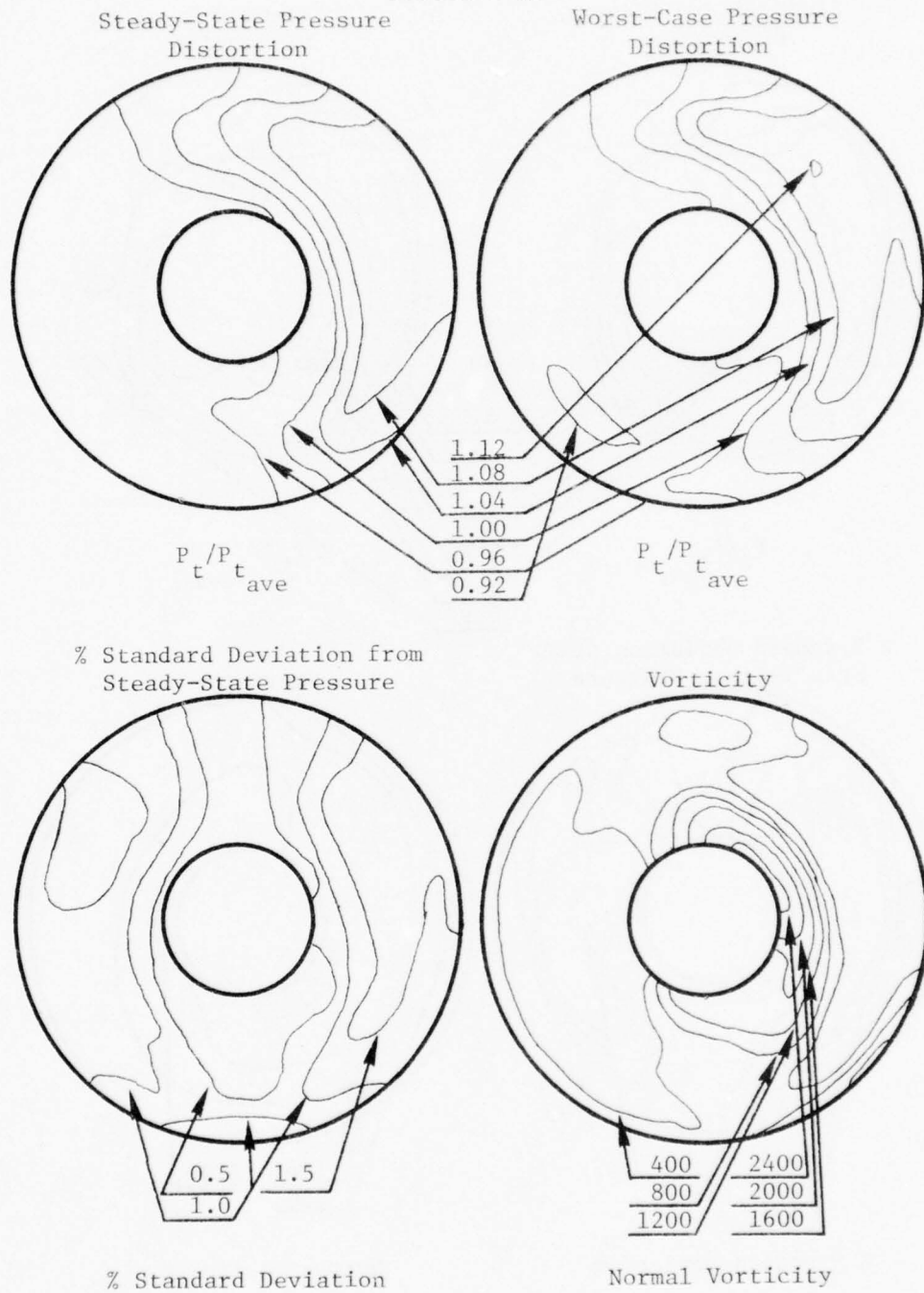


Figure 14

Test HA-14/12 Data Comparison
Screen 1-7F

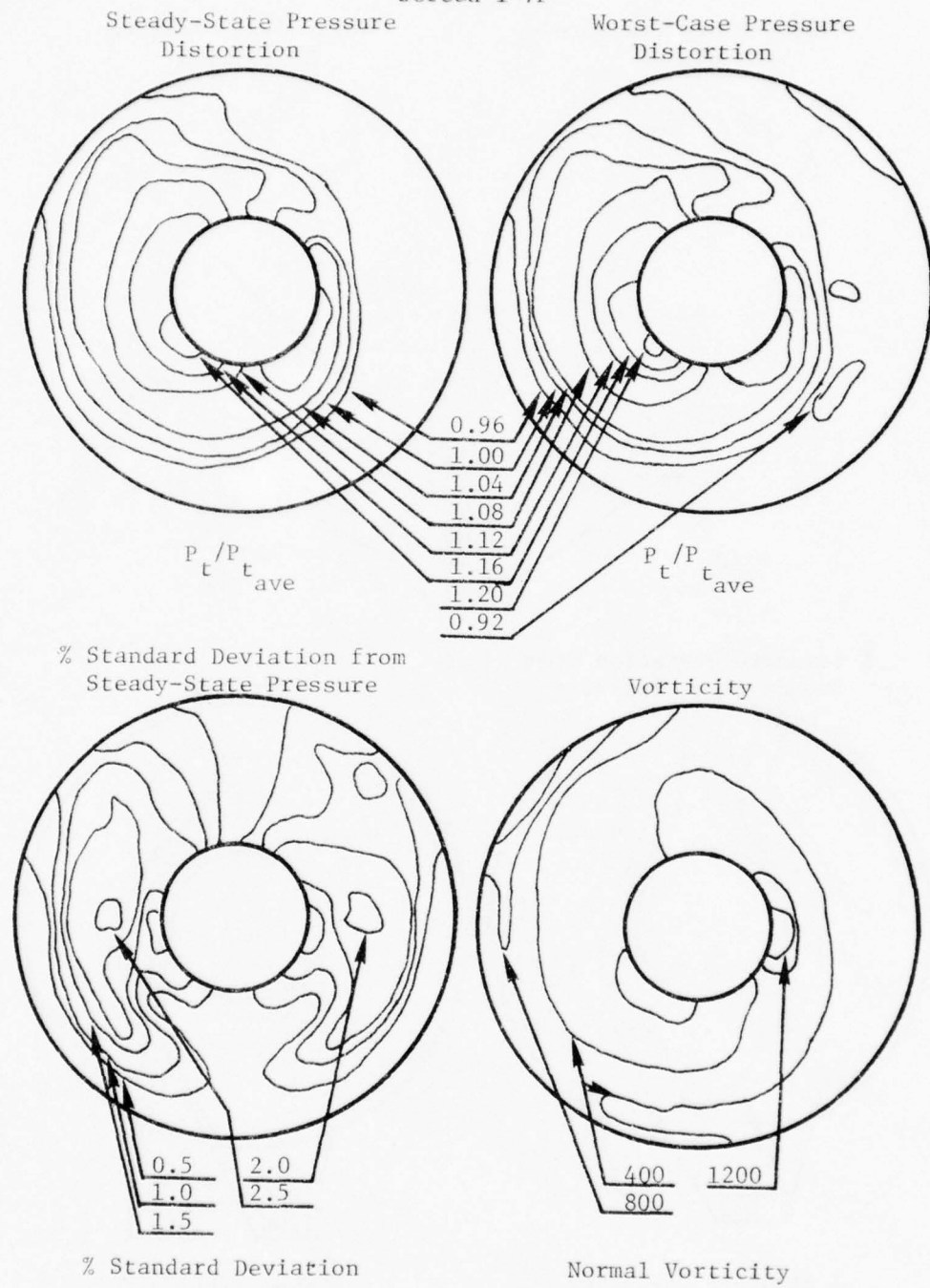


Figure 15

Test HA-18/24 Data Comparison
Screen 3-5F

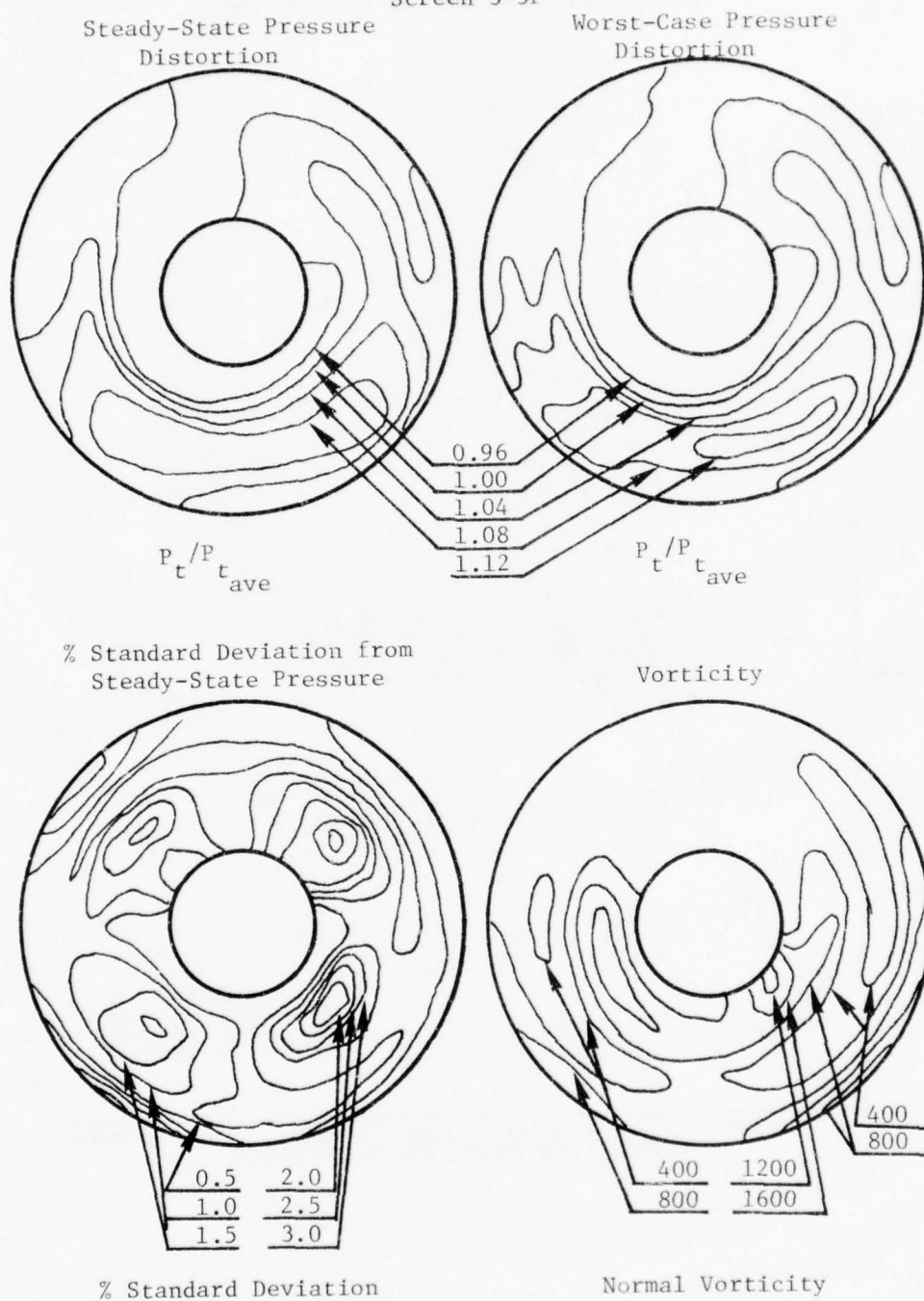


Figure 16

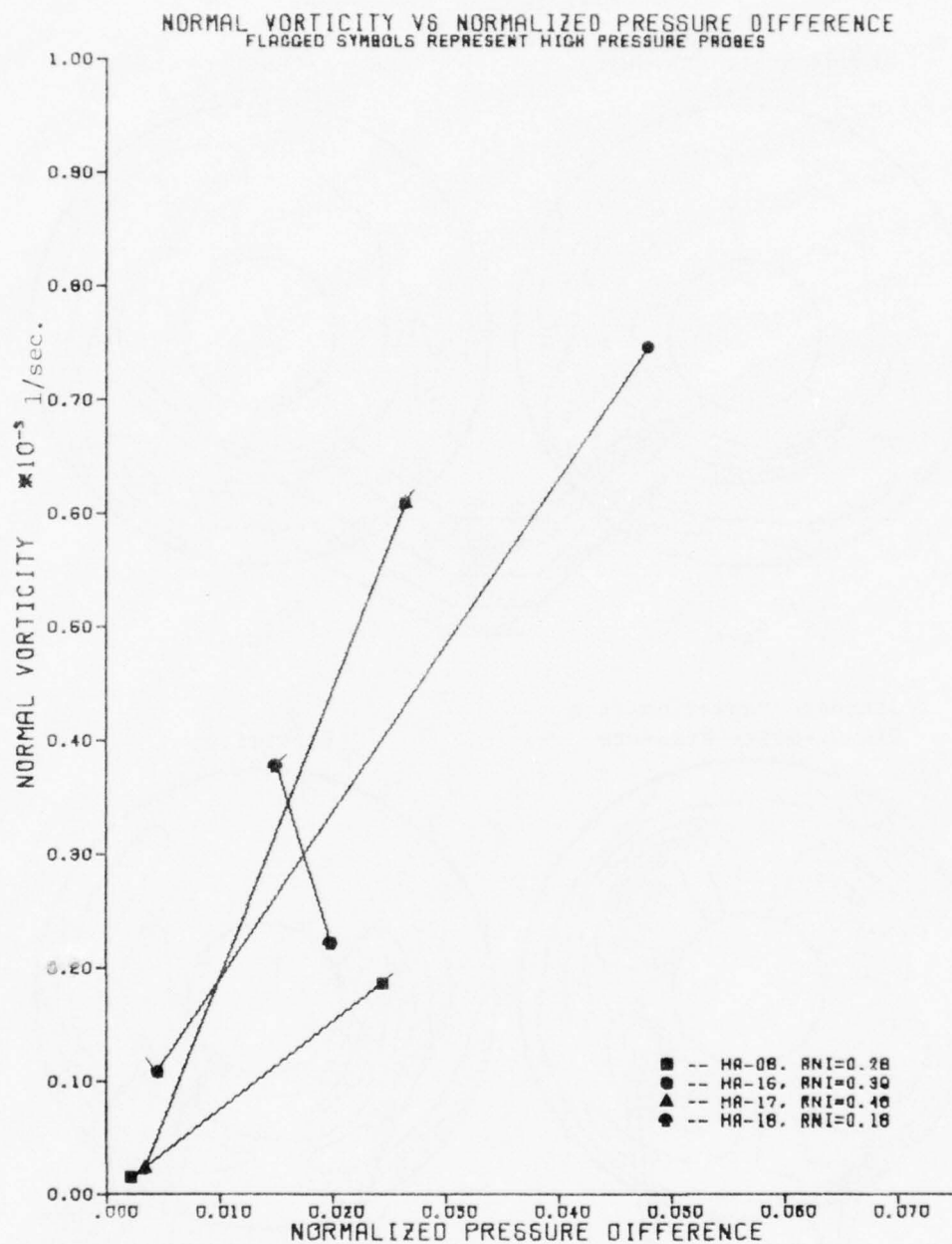


Figure 17

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23. Kimzey, W. F. and McIlveen, M. W., Analysis and Synthesis of Distorted and Unsteady Turbo Engine Inlet Flow Fields, AIAA Paper 71-668, June 1971.

Appendix A

```

PROGRAM VORTEX ( OUTPUT, TAPE5, TAPE6=OUTPUT )
COMMON / SETUP1 / ANGLES(10), RADII(10), TITLE(3), TIME
COMMON / SETUP2 / DELPR, RIN, ROT, NOT, NOA, DELAN, NORAKS, APOPR,
1 COMMON / SETUP3 / GROUP(10,10), GRADTH(10,10)
COMMON / SETUP4 / TT2, PS2, APEA, AIRFLO
COMMON / SETUP5 / PTOPR(10,10), PTOPR(10,10), I2(10,10)
DIMENSION DATA(50), IPROPS(12)
DATA IPROPR, ISCREEN, NPLDT1, TIME, I2 / 2*1., 2, 0., 100*1H

INPUT DEFINITIONS
* = DEFAULT VALUE
IPROPR THE NUMBER OF BAD PRORES IN THE DATA
ISCREEN MAXIMUM NUMBER OF 12
LIST OF THE BAD PRORE NUMBERS, IN ORDER
FOR AUTOMATIC STATISTICAL SCREENING OF DATA
* 0 = NO SCREENING
1 = SCREENING
DELPR FACE PATTERN PRESSURE RATIO INCREMENT
DELWCIR FACE PATTERN CIRCUMFERENTIAL VORTICITY INCREMENT
DELAN FACE PATTERN ANGLE INCREMENT IN DEGREES
RIN FACE PATTERN RADIUS IN INCHES
ROT STATION INNER RADIUS IN INCHES
NOT STATION OUTER RADIUS IN INCHES
NORAKS MINIMUM NUMBER OF PRORES PER RAKE
DATA MINIMUM NUMBER OF 3, MAXIMUM NUMBER OF 10
TITLE NUMBER OF RAKES
RADII ENGINE FACE TOTAL PRESSURES IN PSIA
ANGLES USER INPUT TITLE INFORMATION
PS2 RAKE POSITION IN INCHES
TT2 RAKE POSITIONS IN DEGREES
AIRFLO ENGINE FACE AVERAGE STATIC PRESSURE
NPLDT1 ENGINE FACE AVERAGE TOTAL TEMPERATURE IN DEGREES
METHOD OF DATA CORRECTED AIRFLOW
* 1 = TIME INPUT FROM TAPE
2 = CARQ INPUT
TIME FOR TAPE INPUT, THE TIME OF THE DESIRED DATA

INITIALIZE STATION 2.0 CONDITIONS
DATA RIN, ROT, NOT, NORAKS, DELPR / 5.65, 17.387, 6, 8, 0.74 /
DATA RADII / 7.277, 9.919, 11.992, 13.756, 15.313, 16.736, 4*0. /
DATA ANGLES / 0., 45., 90., 135., 180., 225., 270., 315., 2*0. /

```

CC

```

DATA DELAN, DELVOR, ISWITCH / 5., 400., 1 / DELVOR, PIN,
NAMELIST / DATIN / IPRORE, DELPR, AIRFLO, PACII, TPRORS, ANGLES,
1 ROT, NOT, DATA, TITLE, PACII, TPRORS, ANGLES,
2 PS2, TI2, ISCREEN, NORAKS, TIME, NPLOT1, DELAN
CALL LISTED
NOA = NORAKS + 1
1 READ (5, DATIN)
IF ( EOF(5) ) NE. 0. ) STOP
IF ( TITLE(1) .EQ. 1. ) READ (5, 100) TITLE
100 FORMAT (RA10)

READ DATA
WRITE OUT CONTROL PARAMETERS
WRITE (6, 101) TITLE
101 FORMAT (1H1, //, 1X, RA10)
106 WRITE (6, 106)
106 FORMAT ( //, 23X, "VORTICITY MAP PROGRAM", /, 32X, "A PRODUCT OF",
1 *** CONTROL CARD INPUT ***
2 WRITE (6, 102) TPRORS, ISCREEN, DELPR, DELAN
102 FORMAT ( /, 9X, "IPRORE", 8X, "ISCREEN", 10X, "DELPR", 9X,
1 "DELVOR", 12X, "PIN", /, 2115, 3F15.2, //, 12X, "ROT",
2 12X, "NOT", 9X, "NORAKS", 10X, "DELAN", /, F15.2, 2115,
3 F15.2 )
115 WRITE (6, 115) PS2, TI2, AIRFLO, NPLOT1
115 FORMAT ( /, 12X, "PS2", 12X, "TI2", 9X, "AIRFLO", 9X,
1 "NPLOT1", /, F15.4, F15.2, F15.3, F15.1 )
103 WRITE (6, 103) (ANGLES(I), I=1, NORAKS)
103 FORMAT ( /, 9X, "ANGLES", /, (4F15.2) )
104 WRITE (6, 104) (RADII(I), I=1, NOT)
104 FORMAT ( /, 9X, "RADII", /, (4F15.4) )
105 IF (IPRORE .NE. 0) WRITE (6, 105) (IPRORS(I), I=1, IPRORE)
105 FORMAT ( //, 9X, "DELETED PRORES", /, 10X, 1215,
1 ANGLES(NOAS) = ANGLES(1) + 360.
2 IF ( NPLOT1 .EQ. 2 ) GO TO 7
3 XTIME = TIME
2 CALL SPREAD ( TIME, DATA )
IF ( ( XTIME - :00002 ) .GE. TIME ) GO TO 2
IF ( ( XTIME + :00002 ) .LT. TIME ) GO TO 2
GO TO 5

```



```

3 TIME = TIME + 1.0
CC
PROCESS DATA TO ELIMINATE BAD PRORES
5 IF ( IPRORE .EQ. 0 ) GO TO 7
  DO 5 I=1,48
  DO 5 J=1,IPRORE
  IF ( IPRORS(I) .EQ. I ) DATA(I) = 0.0
5 CONTINUE
  GO TO 8
7 IF ( ISCREEN .EQ. 0 ) GO TO 10
8 CALL SCREEN ( DATA, NOA, NOT, I2, <FAILS )
  IF ( <FAILS .EQ. 1 ) WRITE (6,107) TIME
107 FORMAT ( /, 5X, "A SCAN HAS FAILED SCREEN TEST AT TIME =", F20.5 )
CC
NORMALIZE AND PRINT DATA
13 NUM = 0
  NUMR = 0
  DO 13 I=1,NOA
  DO 13 J=1,NOT
  NUM = NUM + 1
  IF ( I.EQ.NOA ) GO TO 12
  PTOPR(I,J) = DATA(NUM)
  GO TO 13
12 NUMR = NUMR + 1
  PTOPR(I,J) = DATA(NUMR)
  RADII(J) = RADII(J) / ROT
13 CONTINUE
  IF ( ROT .EQ. 1. .AND. ISWITCH .EQ. 1 ) GO TO 14
  ISWITCH = 1
  AREA = ( ROT**2 - RIN**2 ) * 3.1415927
  RIN = RIN / POT
  POT1 = POT
  POT = 1.0
14 CALL COMP(1)
CC
CALCULATE RADIAL VORTICITY
  DO 20 J=1,NOT
  GRADTH(NOA,J) = GRADTH(1,J)
20 CONTINUE
  CALL MACH (XM, GAMMA, A)
  Q = APOPR-PS2

```

VORT0890
VORT0900
VORT0910
VORT0920
VORT0930
VORT0940
VORT0950
VORT0960
VORT0970
VORT0980
VORT0990
VORT1000
VORT1010
VORT1020
VORT1030
VORT1040
VORT1050
VORT1060
VORT1070
VORT1080
VORT1090
VORT1100
VORT1110
VORT1120
VORT1130
VORT1140
VORT1150
VORT1160
VORT1170
VORT1180
VORT1190
VORT1200
VORT1210
VORT1220
VORT1230
VORT1240
VORT1250
VORT1260
VORT1270
VORT1280
VORT1290
VORT1300
VORT1310
VORT1320

VORT11330
VORT11340
VORT11350
VORT11360
VORT11370
VORT11380
VORT11390
VORT11400
VORT11410
VORT11420
VORT11430
VORT11440
VORT11450
VORT11460
VORT11470
VORT11480
VORT11490
VORT11500
VORT11510
VORT11520
VORT11530
VORT11540
VORT11550
VORT11560
VORT11570
VORT11580
VORT11590
VORT11600
VORT11610
VORT11620
VORT11630
VORT11640
VORT11650
VORT11660
VORT11670
VORT11680
VORT11690
VORT11700
VORT11710
VORT11720
VORT11730
VORT11740
VORT11750
VORT11760

```

COEF = A / ( GAMMA * (ROT1/12.) * XM )
DO 21 I=1,NOA
DO 21 J=1,NOT
GRADTH(I,J) = COEF / (PTOPB(I,J)*SQR((PTOPB(I,J)-PS2)/O))
* GRADTH(I,J)
1 CONTINUE
21 CONTINUE
24 CALCULATE CIRCUMFERENTIAL VORTICITY
DO 25 J=1,NOT
GRADR(NOA,J) = GRADR(1,J)
25 CONTINUE
DO 26 I=1,NOA
DO 26 J=1,NOT
GRADR(I,J) = -COEF / (PTOPB(I,J)*SQR((PTOPB(I,J)-PS2)/O))
* GRADR(I,J)
1 CONTINUE
26 CONTINUE
27 WRITE OUT VORTICITY VALUES
WRITE (6,101) TITLE
WRITE (6,108) TIME
108 FORMAT ( /, 10X, "CASE NUMBER =", F10.5 )
WRITE (6,110)
110 FORMAT ( /, 1X, "RADIAL VORTICITY ***")
WRITE (6,111) ( ANGLES(I), I=1,NORAKS)
111 FORMAT ( /, 6X, "RAKE =", F14.1, /, 6X, "RD / OD")
DO 27 J=1,NOT
WRITE (6,113) RADII(J), ( GRADTH(I,J), I=1,NORAKS )
113 FORMAT ( F11.5, 2X, F14.2 )
113 CONTINUE
WRITE (6,114)
114 FORMAT ( /, 1X, "CIRCUMFERENTIAL VORTICITY ***")
WRITE (6,111) ( ANGLES(I), I=1,NORAKS)
DO 28 J=1,NOT
WRITE (6,113) RADII(J), ( GRADR(I,J), I=1,NORAKS)
28 CONTINUE
DO 29 I=1,NORAKS
DO 29 J=1,NOT
PTOPR(I,J) = SQR ( GRADR(I,J)**2 + GRADTH(I,J)**2 )
29 CONTINUE
WRITE (6,115)
DO 31 J=1,NOT
PTOPR(NOA,J) = PTOPR(1,J)

```

```

31 CONTINUE
115 FORMAT ( //, 1X, '*** TOTAL VORTICITY ***' )
WRITE (6,111) (ANGLES(I),I=1,NORAKS)
DO 30 J=1,NORAKS
WRITE (6,113) RADII(J), (PTOPR(I,J),I=1,NORAKS)
30 CONTINUE

CC PRINT OUT MAP OF TOTAL VORTICITY

CALL COMP(2)
GO TO 1
END
SUBROUTINE BMFIT(L,N,X,Y,SLOPE0,SLOPEN,A,B,C,D)
DIMENSION X(150),Y(150),A(150),B(150),C(150),D(150)
KC = (L-1)/3 - 2*L
DO 1 J=2,N
A(J) = X(J)-X(J-1)
1 D(J) = (Y(J)-Y(J-1))/A(J)
K2 = N-2
IF (KN-1) 4,5,6
4 C(N) = 3.0*(SLOPEN-D(N))/A(N)
C(N) = 0.5
K3 = N
K1 = 1
GO TO 8
5 T = 2.0*(A(N)+A(N-1))
B(N) = SLOPEN
GO TO 7
6 T = 3.0*A(N)+A(N-1)+A(N-1)
SLOPEN = 0.0
7 C(N-1) = (6.0*(D(N)-D(N-1))-SLOPEN*A(N))/T
B(N-1) = A(N-1)/T
K3 = N-1
K1 = 2
DO 9 J = K1,K2
K = N-J
T = 2.0*(A(K)+A(K+1))-A(K+1)*B(K+1)
B(K) = A(K)/T
9 C(K) = (6.0*(D(K+1)-D(K))-A(K+1)*C(K+1))/T
IF (K0-1) 12,13,11
11 B(1) = (6.0*(D(2)-SLOPE0)-C(2)*A(2))/(A(2)*(2.0-B(2)))
K1 = 2
GO TO 15

```

```

VORT11770
VORT11780
VORT11790
VORT11800
VORT11810
VORT11820
VORT11830
VORT11840
VORT11850
VORT11860
VORT11870
VORT11880
BMFI0010
BMFI0020
BMFI0030
BMFI0040
BMFI0050
BMFI0060
BMFI0070
BMFI0080
BMFI0090
BMFI0100
BMFI0110
BMFI0120
BMFI0130
BMFI0140
BMFI0150
BMFI0160
BMFI0170
BMFI0180
BMFI0190
BMFI0200
BMFI0210
BMFI0220
BMFI0230
BMFI0240
BMFI0250
BMFI0260
BMFI0270
BMFI0280
BMFI0290
BMFI0300
BMFI0310
BMFI0320

```

```

12 B(1) = SLOPE0
13 B(2) = C(2)-B(2)*R(1)
14 GO TO 14
15 B(1) = (6.0*(O(3)-J(2))-A(3)*C(3))/(3.0*A(2)+A(3))*(2.0-B(3))
16 B(2) = R(1)
17 K1 = 3
18 J = K1,K3
19 R(J) = C(J)-R(J)*B(J-1)
20 IF (KN-1) 18,18,17
21 B(N) = B(N-1)
22 CURIC COEFFICIENTS
23 DO 19 J=2,N
24 TA = 6.0 * A(J)
25 TM = X(J-1)*X(J)
26 TN = X(J-1)*B(J)
27 TX = X(J)*X(J)-X(J-1)-X(J-1)
28 TY = X(J-1)*(X(J)+X(J-1))/TA
29 A(J-1) = (R(J)-B(J-1))/TA
30 C(J-1) = O(J)+(R(J)*(-TX)+2.0*X(J-1)*TN-R(J-1)*TM)/TA
31 B(J-1) = (TM-TN)*3.0/TA
32 O(J-1) = (TX*TN+TY*TM)/TA +Y(J-1)-X(J-1)*D(J)
33 RETURN
34 END
35 SURROUTINE RMVAL(NV, XINP, M, VALU, A2, B2, C2, D2, IO, 7, XDERV1)
36 DIMENSION
37 1 A2 XINP (10,15), B2 (10,15), C2 (10,15), D2 (10,15),
38 2 XINP (100)
39 M = 0
40 DO 1 J=1,NN
41 IF (XINP(J)-VALU) 1,3,3
42 CONTINUE
43 OUT OF RANGE
44 M = 1
45 J = NN
46 GO TO 4
47 IF (J-1) 5,5,4
48 Z = VALU*(VALU*A2(J-1,IO)+R2(J-1,IO))+C2(J-1,IO)+D2(J-1,IO)
49 XDERV1 = VALU*(3.0*VALU*A2(J-1,IO)+2.0*R2(J-1,IO))+C2(J-1,IO)
50 RETURN
51 J = J+1
52 GO TO 4
53 SURROUTINE CHECK ( I )
54 GO TO ( 3, 3, 3, 4, 5, 6, 10, 10, 10, 10 ), I
55
RMFI0330
RMFI0340
RMFI0350
RMFI0360
RMFI0370
RMFI0380
RMFI0390
RMFI0400
RMFI0410
RMFI0420
RMFI0430
RMFI0440
RMFI0450
RMFI0460
RMFI0470
RMFI0480
RMFI0490
RMFI0500
RMFI0510
RMFI0520
RMFI0530
RMFI0540
RMFI0550
RMVA0010
RMVA0020
RMVA0030
RMVA0040
RMVA0050
RMVA0060
RMVA0070
RMVA0080
RMVA0090
RMVA0100
RMVA0110
RMVA0120
RMVA0130
RMVA0140
RMVA0150
RMVA0160
RMVA0170
RMVA0180
RMVA0190
CHEC0010
CHEC0020

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```

301 WRITE(6,301)
   FORMAT(5X,51H JN INDEX OF PRES VARIABLE EXCEEDS 12
   RETURN
401 WRITE(6,401)
   FORMAT(5X,51H JO INDEX OF ANGFI VARIABLE EXCEEDS 120
   RETURN
501 WRITE(6,501)
   FORMAT(5X,51H JP INDEX OF ANG VARIABLE EXCEEDS 30
   RETURN
601 WRITE(6,601)
   FORMAT(5X,51H IN INDEX OF PRF1,PA,2ADD AND YANG EXCEEDS 120
   RETURN
1001 WRITE(6,1001)
   FORMAT(5X,51H TOTAL NUMBER OF POINTS IN PATTERN EXCEEDS 840
   RETURN
END
SUBROUTINE COMP(ITEST)
COMMON / SETUP1 / ANG(10), RAD(10), TITLE(8), TIME
COMMON / SETUP2 / DELPB, RIN, ROT, NOI, NOA, DELAN, NORAKS, APOPR,
1 COMMON / SETUP5 / PTOPR(10,10), PTOB3(10,10), I2(10,10)
DIMENSION AJOAT(10), AVGPBI(20), C(21), CC(21,10), PPAOPR(10),
1 SUM(20), TOTPC(10), X(11), XC(21,11), Y(21,10), YY(21),
2 SINGAV(10)
DATA YC, XC / 232*0.1 /
DATA ISAVE / 0 /
DELR = (1.0 - RIN)/10.0
NPHI = NOA
RADIAN = .017453292
PI = 3.1415927
TWOPI = 2.0 * PI
DO 906 I = 1, NOA
X(I+1) = ANG(I) * RADIAN
X(1) = X(NOA) - TWOPI
X(NOA+2) = X(3) + TWOPI
DO 907 J = 1, NOT
Y(1,J) = PTOB3(NPHI-1, J)
DO 908 I = 1, NPHI
N=I+1
Y(N,J)=PTOPR(I,J)
907 Y(NPHI+2, J) = PTOPR(2,J)
JFLAG = 0
JFLAG = 0
NPIP=NPHI+2

```

CHEC00070
 CHEC00040
 CHEC00050
 CHEC00060
 CHEC00070
 CHEC00080
 CHEC00090
 CHEC00100
 CHEC00110
 CHEC00120
 CHEC00130
 CHEC00140
 CHEC00150
 CHEC00160
 CHEC00170
 CHEC00180
 COMP0010
 COMP0020
 COMP0030
 COMP0040
 COMP0050
 COMP0060
 COMP0070
 COMP0080
 COMP0090
 COMP0100
 COMP0110
 COMP0120
 COMP0130
 COMP0140
 COMP0150
 COMP0160
 COMP0170
 COMP0180
 COMP0190
 COMP0200
 COMP0210
 COMP0220
 COMP0230
 COMP0240
 COMP0250
 COMP0260
 COMP0270
 COMP0280

COMP00290
COMP00300
COMP00310
COMP00320
COMP00330
COMP00340
COMP00350
COMP00360
COMP00370
COMP00380
COMP00390
COMP00400
COMP00410
COMP00420
COMP00430
COMP00440
COMP00450
COMP00460
COMP00470
COMP00480
COMP00490
COMP00500
COMP00510
COMP00520
COMP00530
COMP00540
COMP00550
COMP00560
COMP00570
COMP00580
COMP00590
COMP00600
COMP00610
COMP00620
COMP00630
COMP00640
COMP00650
COMP00660
COMP00670
COMP00680
COMP00690
COMP00700
COMP00710
COMP00720

```

DO 909 J = 1, NOT
  IFLAG = 0
  KFLAG = 0
  LL = 0
  DO 910 I=1,NPTP
    VV(I)=V(I,J)
    CALL SPL503 ( X, VV, NPIP, C )
  DO 911 I=1,NPIP
    CC(I,J)=C(I)
  DO 912 I=1,NPHI
    H=X(I+1)-X(I)
    DELTA=H/10.
    XP=X(I)
    IF ( I .GT. 2 .OR. V(I,J) .GE. PRAOPR(J) ) GO TO 954
    IFLAG = 1
    ISAVE = 1
    KFLAG = 1
    DO 913 N=XP+DELTA
      ETA=(XC(I,N)-X(I))/H
      PT1=Y(I,J)+(V(I+1,J)-V(I,J))*ETA-CC(I,J)*(4**2)**(ETA-ETA**2)
      PT2=H**2*(CC(I,J)-CC(I+1,J))*(3*ETA**3 -5*ETA**4+2*ETA**5)
      YC = PT1 + PT2
      IF ( IFLAG .EQ. 0 .AND. YC .GT. PRAOPR(J) .OR.
        1 THETA1=(PRAOPR(J)-YC) / (YC-YC1) * (XC(I,N)-XP) + XP
      IF (IFLAG .EQ. 0 ) GO TO 956
      PRMIN = VV(ISAVE)
      LIM = I + 1
      IF (VY(L) = ISAVE,LIM) PRMIN = VV(L)
      IF (VY(L) .LT. PRMIN)
        954 CONTINUE
      IF ( PRMIN .GE. PRAOPR(J) ) GO TO 958
      IF ( KFLAG .EQ. 0 .OR. ISAVE .NE. 2 ) GO TO 959
      THETS = THETA1 - X(2)
      PPSAVE = PRMIN
      CALL DIF25 ( THETA1, THETA2, X(2), KFLAG, J, THETS, LL, ITROLE )
      GO TO 958
      ISAVE = I
      THETA2 = THETA1
      IFLAG = 1
      GO TO 955

```

```

959 IF (KFLAG.EQ.1 .AND. I.EQ.NPHI .AND. N.EQ.11) GO TO 957
CALL DIF25 ( THETA1, THETA2, X(2), KFLAG, J, THETS, LL, ITPOLE )
IF ( ITPOLE .EQ. 1 ) GO TO 958
GO TO 961
957 IF ( PRSAVE .LT. PRMIN ) PRMIN = PRSAVE
KFLAG = 0
961 LL = LL + 1
958 IFLAG = 0
955 IF (KFLAG .EQ. 1 .AND. I .EQ. NPHI .AND. N .EQ. 11) GO TO 962
YCI = VC
XP = XC(I,N)
913 CONTINUE
912 CONTINUE
IF (JFLAG .EQ. 1) JFLAG = 0
GO TO 909
904 CONTINUE
DO 914 I = 1, NORAKS
AVGPHI(I) = ( ANG(I+1) + ANG(I) ) / 2.0
SUM(J) = 0.0
9917 I=2,NPHI
IF (I.NE.NPHI) GO TO 99917
SUM(J) = SUM(J) + TOPR(I,J) * (AVGPHI(1) + (360. - AVGPHI(NORAKS))) / 360.
99917 SUM(J) = SUM(J) + TOPR(I,J) * (AVGPHI(I) - AVGPHI(I-1)) / 360.
9917 CONTINUE
PRAOPR(J) = SUM(J)
JFLAG = 1
GO TO 905
909 CONTINUE
ATOTAL = ( 1.0 - RIN**2 ) * PI / 4.0
TOTPC(J) = ( RAD(J)**2 - RIN**2 ) * PI / ( 4.0 * ATOTAL )
IF (TOTPC(J)) 923, 922, 922
923 WRITE (6,925) J, RAD(J), RIN
925 FORMAT (1X,11HAT, PING NO.,I2,16H THE INPUT DIAM.,F8.4,22H IS LESS
1 THAN ID DIAM.,F8.4/)
GO TO 821
922 CONTINUE
AJOAT(1) = (TOTPC(1) + TOTPC(2)) / 2.
K = NOT 1
DO 930 J=2,K
AJOAT(J) = (TOTPC(J+1) - TOTPC(J-1)) / 2.
AJOAT(NOT) = 1.0 - ( TOTPC(NOT-1) + TOTPC(NOT) ) / 2.0
APOPR=0.0

```

COMP0730
 COMP0740
 COMP0750
 COMP0760
 COMP0770
 COMP0780
 COMP0790
 COMP0800
 COMP0810
 COMP0820
 COMP0830
 COMP0840
 COMP0850
 COMP0860
 COMP0870
 COMP0880
 COMP0890
 COMP0900
 COMP0910
 COMP0920
 COMP0930
 COMP0940
 COMP0950
 COMP0960
 COMP0970
 COMP0980
 COMP0990
 COMP1000
 COMP1010
 COMP1020
 COMP1030
 COMP1040
 COMP1050
 COMP1060
 COMP1070
 COMP1080
 COMP1090
 COMP1100
 COMP1110
 COMP1120
 COMP1130
 COMP1140
 COMP1150
 COMP1160

```

931 DO 931 J = 1, NOT
    AOPR=APOPR+PRAOPR(J)*AJOAT(J)
    IF ( ITEST.EQ. 1 ) GO TO 933
    DO 932 I=1,NOA
    DO 932 J=1,NOT
        PTOPR(I,J) = PTOPR(I,J)
932 CONTINUE
    GO TO 821
933 DO 32 J=1,NOT
    DO 32 I=1,NOA
        PTOPR(I,J) = PTOPR(I,J)/AOPR
32 CONTINUE
    IF ( ITEST.EQ.2 ) GO TO 821

    CALCULATE RING AVERAGE PRESSURE

    DO 2 J=1,NOT
        RINGAV(J) = 0.0
    DO 1 I=1,NORAKS
        RINGAV(J) = RINGAV(J) + PTOPR(I,J)
1 CONTINUE
    RINGAV(J) = RINGAV(J) / FLOAT(NORAKS)
2 CONTINUE
    CALL PREPAT ( DELR, IM, K, ITEST )
    CALL PATERN(K, ITEST)
    RETURN
    END
    SUBROUTINE CONPUN ( JN, RADR, JO, PRESS, ANGFI, 32, C2, C2, C2, A2,
        PRS, BRL, IM, K, ITEST )
        JSIM(R40), PLABEL(36), XTEM(840), YTEM(840),
        MAXSYM
1 COMMON / SETUP2 / DELPR, PIN, ROT, NOT, NOA, DELAN, NORAKS, AOPR,
        DELVOR
1 COMMON / SETUP5 / PTOPR(10,10), PTOPR(10,10), I2(10,10)
    DIMENSION ANGFI(1), R2(10,15), C2(10,15), PRF1(120),
1 A2(10,15), PRESS(120,12), PRP(12), RADN(120), SANG(1), YANG(120)
    MAXSYM = 1
    DO 5 IZ = 1, 840
        XTEM(IZ) = 0.0
        YTEM(IZ) = 0.0
        JSIM(IZ) = 0
5 ING = 0
    K = 0

```

```

COMB1170
COMB1180
COMB1190
COMB1200
COMB1210
COMB1220
COMB1230
COMB1240
COMB1250
COMB1260
COMB1270
COMB1280
COMB1290
COMB1300
COMB1310
COMB1320
COMB1330
COMB1340
COMB1350
COMB1360
COMB1370
COMB1380
COMB1390
COMB1400
COMB1410
COMB1420
COMB1430
COMB1440
COMB1450
COMB1460
COMB1470
COMB1480
COMB1490
COMB1500
COMB1510
COMB1520
COMB1530
COMB1540
COMB1550
COMB1560
COMB1570
COMB1580
COMB1590
COMB1600
COMB1610
COMB1620
COMB1630
COMB1640
COMB1650
COMB1660
COMB1670
COMB1680
COMB1690
COMB1700
COMB1710
COMB1720
COMB1730
COMB1740
COMB1750
COMB1760
COMB1770
COMB1780
COMB1790
COMB1800
COMB1810
COMB1820
COMB1830
COMB1840
COMB1850
COMB1860
COMB1870
COMB1880
COMB1890
COMB1900
COMB1910
COMB1920
COMB1930
COMB1940
COMB1950
COMB1960
COMB1970
COMB1980
COMB1990
COMB2000

```

```

NNING = NING
IF ( ITEST.EQ. 1 ) DEL = DELPR
IF ( ITEST.EQ. 2 ) DEL = DELVOR
DO 393 MX=1,J0
DO 32 J=1,JN
PRP(J) = PRESS(MX,J)
CALL RMFIT (5,JN,PAOR(1),POP(1),0.0,0.0,A2(1,1),32(1,1),C2(1,1),D2
1(1,1))
CALL FALPO ( PRP(1), IM, PRS, DEL, M, PRF1, 7, 32, TEST, D2,
1
IF ( IM.EQ. 0 ) GO TO 393
IF ( IM.LE.120 ) GO TO 34
CALL CHECK (6)
GO TO 40
PRP = PRS
34
ING = 0
NNING = NNING
DO 392 J=1,IM
K = K + 1
YANG(J) = RADD(J) * COS ( ANGFI(MX) * 0.0174533 )
RADD(J) = RADD(J) * SIN ( ANGFI(MX) * 0.0174533 )
IF ( TEST.NE.0 ) GO TO 40
IF ( PRP - PRF1(J) ) 37,39,36
IF ( PRF1(J)-1.0 ) 37,39,392
PRP = PRP + DEL
IF ( ING.GE.9 ) GO TO 381
ING = ING + 1
GO TO 35
IF (ING-16) 382,383,384
381
382
ING = ING + 7
GO TO 35
383
ING = ING + 16
GO TO 35
384
ING = 0
NNING = NNING + 1
GO TO 35
39
CALL OUTP ( NING, PRF1, K, ING, PLAREL, JSTM, MAXSYM, J )
IF ( K.GT.840 ) RETURN
ITEM(K) = RADD(J)
ITEM(K) = YANG(J)
392
393
40
CONTINUE
RETURN
END

```

CONP0180
 CONP0190
 CONP0200
 CONP0210
 CONP0220
 CONP0230
 CONP0240
 CONP0250
 CONP0260
 CONP0270
 CONP0280
 CONP0290
 CONP0300
 CONP0310
 CONP0320
 CONP0330
 CONP0340
 CONP0350
 CONP0360
 CONP0370
 CONP0380
 CONP0390
 CONP0400
 CONP0410
 CONP0420
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 CONP0450
 CONP0460
 CONP0470
 CONP0480
 CONP0490
 CONP0500
 CONP0510
 CONP0520
 CONP0530
 CONP0540
 CONP0550
 CONP0560
 CONP0570
 CONP0580
 CONP0590
 CONP0600
 CONP0610

```

SUBROUTINE DIF25 ( THETA1, THETA2, X2, KFLAG, NOA, THETS, LL,
1 ITROLE )
DATA ISAVNO / 0 /
ITROLE = 0
IF ( NOA .EQ. ISAVNO ) GO TO 10
SAVET1 = THETA1 + 0.4363323
SAVET2 = THETA2
SAVEX2 = X2
ISAVFL = KFLAG
ISAVNO = NOA
RETURN
10 IF ( THETA2 .GT. SAVET1 ) RETURN
LL = LL - 1
THETA2 = SAVET2
IF ( ISAVFL .EQ. 0 ) RETURN
LL = LL + 1
ITROLE = 1
THETS = THETA1 - SAVEX2
RETURN
END
SUBROUTINE FALPO (PRP,IM,PRS,DELPR,M,PRF1,7,32,TEST,02,NOUNT,A2,C2,
1 JN,PRL,RADD,RADR)
DIMENSION
1 PRP (12 ) ; PRF1 (120 ) ; RADJ (120 ) ; RAPP (12 ,15 ) ;
2 A2 (10,15 ) ; B2 (10,15) ; C2 (10,15) ; D2 (10,15 ) ;
IM = 1
PRF1(IM) = PRS
NOUNT = 0
JM = JN - 1
DO 29 JK=1,JM
NOUNT = 0
IF (PRP(JK)-PRF1(IM)) 25,27,28
IF (PRP(JK+1)-PRF1(IM)) 29,29,26
IF (PRP(JK+1)-PRF1(IM)) 26,29,29
RADLO = RADP(JK)
RADHI = RADP(JK+1)
PRELO = PRP(JK)
PREHI = PRP(JK+1)
VALU = (RADHI+RADLO)/2.0
CALL RMVAL (JN, RADR(1), M, VALU, A2(1,1), B2(1,1), C2(1,1),
5 D2(1,1), 1, 7, XDERV1)
IF (ABS(Z-PRF1(IM))-0.005) 7,7,530
1 IF (Z-PRF1(IM)) 3,7,1
IF (PRELO - PRF1(IM)) 2,4,9

```


X DATA	-0.1396	/	0.023400	0.3100	0.1700	PATE0330
XXXXXX	0.2700		0.2000	0.3100	0.1700	PATE0340
XXXXXX	0.1700		0.0600	0.0600	0.0600	PATE0350
XXXXXX	0.0600		0.0600	0.0600	0.0600	PATE0360
XXXXXX	0.2000		0.1700	0.1700	0.1700	PATE0370
XXXXXX	0.3100		0.3100	0.3100	0.3100	PATE0380
DATA	0.3919	/	0.2800	0.3919	0.3919	PATE0390
XXXXXX	0.57249		0.5103	0.5103	0.5103	PATE0400
XXXXXX	0.7684		0.6432	0.6432	0.6432	PATE0410
XXXXXX	0.8000		0.7945	0.7945	0.7945	PATE0420
XXXXXX	0.8773		0.8660	0.8660	0.8660	PATE0430
XXXXXX	0.9165		0.8370	0.8370	0.8370	PATE0440
XXXXXX	0.9704		0.9655	0.9655	0.9655	PATE0450
XXXXXX	0.9887		0.9958	0.9958	0.9958	PATE0460
XXXXXX	1.0050		0.9982	0.9982	0.9982	PATE0470
XXXXXX	0.9637		0.9718	0.9718	0.9718	PATE0480
XXXXXX	0.9474		0.9539	0.9539	0.9539	PATE0490
XXXXXX	0.8980		0.8879	0.8879	0.8879	PATE0500
DATA	0.8417	/	0.8145	0.8145	0.8145	PATE0510
XXXXXX	0.8000		0.7845	0.7845	0.7845	PATE0520
XXXXXX	0.7684		0.7499	0.7499	0.7499	PATE0530
XXXXXX	0.6572		0.6250	0.6250	0.6250	PATE0540
XXXXXX	0.3919	/	0.3800	0.3800	0.3800	PATE0550
DATA	0.9200		0.9600	0.9600	0.9600	PATE0560
XXXXXX	0.7600		0.7800	0.7800	0.7800	PATE0570
XXXXXX	0.6400		0.5200	0.5200	0.5200	PATE0580
XXXXXX	0.5400		0.4400	0.4400	0.4400	PATE0590
XXXXXX	0.4800		0.3600	0.3600	0.3600	PATE0600
XXXXXX	0.4000		0.2600	0.2600	0.2600	PATE0610
XXXXXX	0.3300		0.1400	0.1400	0.1400	PATE0620
XXXXXX	0.2700		0.0600	0.0600	0.0600	PATE0630
XXXXXX	0.2000		0.0000	0.0000	0.0000	PATE0640
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0650
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0660
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0670
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0680
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0690
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0700
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0710
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0720
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0730
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0740
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0750
XXXXXX	0.1700		0.0000	0.0000	0.0000	PATE0760

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 -0.8600
 -0.9200

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 -0.9200

DATA /
 DUMMY(1) = AOPR
 DUMMY(2) = TIME
 DUMMY(3) = TT2
 DUMMY(4) = PS2
 DUMMY(5) = AIRFLO
 LADD = 0
 IF (ITEST.EQ. 2) LADD = 3
 DO 330 I=1,1000
 IRAY(I)=0
 NO 331 J=1,95
 JRAY(J) = IBLANK
 DO 6 IX = 1, K
 IA = JSIM (IX)
 JSIM (IX) = A (IA)
 CONTINUE
 KI = K
 DO 1 I = 1, 32
 YTEM(I + KI) = YIN(I)
 XTEM(I + KI) = XIN(I)
 JSIM(I + KI) = JUTSIN
 CONTINUE
 KI = KI + 32
 DO 2 J = 1, 94
 YTEM(J + KI) = YOUT (J)
 XTEM(J + KI) = XOUT (J)
 JSIM(J + KI) = JUTSIN
 CONTINUE
 KI = KI + 94

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3      J = 1, 34
      YTEM ( J + KI ) = YOUU ( J )
      XTEM ( J + KI ) = XOUU ( J )
      JSIM (J+KI)=JUTSIN
      CONTINUE
      KI = KI + 34
      JTOT = KI + NOT * NORAKS
      CALL SORT1 ( JSIM , YTEM, XTEM, JTOT )
      DO 30 J = 1, JTOT
      IRAY(J) = IFIX(YTEM(J) * 28. + SIGN (.5, YTEM(J))) + 29
      CONTINUE
      WRITE (6,75) ESCORD
      FORMAT (*1*11X,9A10,/)
      DO 35 J = 1, 95
      IRAY(K1) = IRLANK
      CONTINUE
      DO 40 K1 = 1, JTOT
      IFAY (K1) = 58 - IRAY ( K1)
      LINES = 1
      JIP = 1
      IF ( IRAY(JIP) .GT. LINES ) GO TO 70
      ITEMP = IFIX ( YTEM(JIP) * 47.0 + SIGN ( 0.5, XTEM(JIP) ) ) + 48
      IRAY(ITEMP) = JSIM(JIP)
      IF ( JIP .EQ. JTOT ) GO TO 70
      JIP = JIP + 1
      GO TO 50
      IF ( LINES .GT. (2+LADD) ) GO TO 77
      IF ( LINES .EQ. 2 ) GO TO 71
      IF ( LINES .EQ. 1 .AND. ITEMP .EQ. 2 ) GO TO 74
      WRITE (6,201) IRAY, ( TEMP(LINES,J), J=1,3 ), NUNMY(LINES)
      FORMAT (1X,95A1,6X,3A4,F14.5)
      GO TO 78
      WRITE (6,201) IRAY, ( VORT(LINES,J), J=1,3 ), NUNMY(LINES)
      GO TO 78
      IF ( LINES .EQ. 29 ) IRAY(43) = JUTSIN
      IF ( LINES .GT. (MAXSYM+4+LADD) ) GO TO 74
      IF ( LINES .EQ. (LADD+3) ) GO TO 74
      IF ( LINES .EQ. (LADD+4) ) GO TO 75
      WRITE (6,795) IRAY, A(LINES-4-LADD), PLABEL(LINES-4-LADD)
      FORMAT (1X, 95A1, 11X, A2, 9X, F10.5)
      GO TO 78
      WRITE (6,800) IRAY
      FORMAT (1X,95A1)
      GO TO 78

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```

75 IF ( ITEST.EQ. 2 ) GO TO 76
WRITE (6,795) JRAY, R
GO TO 78
76 WRITE (6,795) JRAY, C
795 FORMAT (1X,95A1,3X,8A4)
79 LINES = LINES + 1
DO 80 K2 = 1, 95
80 JRAY(K2) = IRLANK
IF ( LINES.LT. 58 ) GO TO 50
RETURN
END
SUBROUTINE PREPAT (DELR,IM,K,ITEST)
COMMON / SETUP1 / ANG(10), RAN(10), TITLE(9), TIME
COMMON / SETUP2 / DELPR, RIN, POT, NOT, NOA, DELAN, NORAKS, APOPR,
1 COMMON / SETUP3 / GRAOR(10,10), GADTH(10,10)
COMMON / SETUP5 / PTOPR(10,10), PTOPB(10,10), T2(10,10)
COMMON / A(10), A2(10,15), ANGEI(120), B2(10,15), C2(10,15),
1 DIMENSION D(10), D2(10,15), PRES(15,12), PDS(12), Q(10), PADS(10),
2 S(10), XPSAV(15,3), PRESS(120,12)
000 CALCULATE THE MAX AND MIN NORMALIZED PRESSURES
DELTH = DELAN / 360. * 2.0 * 3.1415927
PRL = PTOPR(1,1)
PRS = PRL
DO 5100 I=1,NOA
DO 5300 J=1,NOT
IF ( PTOPR(I,J) .GT. PRL ) PRL = PTOPR(I,J)
IF ( PTOPR(I,J) .LT. PRS ) PRS = PTOPR(I,J)
5901 CONTINUE
5100 DISMAX = PRL - PRS
DISBAR = 1.0 - PRS
IF (ITEST.EQ.2) GO TO 6200
CALL PRINT ( PRS, PRL, DISMAX, DISBAR)
5200 PRST = 1.0
5300 PRST = PRST-DELPP
5400 IF (PRST-PRS) 6400,6500,6300
5500 PRS = PRST+DELPP
6400 PRST = 0
6500 KNOT1 = 0
000 CALCULATE INTERPOLATED PRESSURES FOR PATTERN USING POLYNOMIAL
CURVE FIT

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C
IF ( NOT .NE. 3 ) GO TO 6550
KNOT1 = 1
DO 6520 I=1,NOA
  XPRSAV(I,3) = PIOP3(I,3)
  PIOP3(I,5) = PIOP3(I,3)
  XPRSAV(I,2) = PIOP3(I,2)
  PIOP3(I,3) = PIOP3(I,2)
  PIOP3(I,4) = ( PIOP3(I,5) + PIOP3(I,3) ) / 2.0
  PIOP3(I,2) = ( PIOP3(I,3) + PIOP3(I,1) ) / 2.0
  XPRSA3 = RAD(3)
  XPRSA2 = RAD(2)
  RAD(5) = RAD(3)
  RAD(3) = RAD(2)
  RAD(4) = ( RAD(5) + RAD(3) ) / 2.0
  RAD(2) = ( RAD(1) + RAD(3) ) / 2.0
  NOT = 5
6520 CONTINUE
DO 6600 I=1,NOA
  J=1,NOT
  PRP(J) = PIOP3(I,J)
  CALL BMFIT (5,NOT,RAD(1),PRP(1),0.0,0.0,A(1),0(1),S(1),0(1))
  DO 6700 JK=1,NOT
    A2(JK,I) = A(JK)
    C2(JK,I) = S(JK)
    Q2(JK,I) = Q(JK)
    O2(JK,I) = O(JK)
  6700 CONTINUE
  5800 IF ( ITEST.E0.2 ) GO TO 6901
  6550 CONTINUE
  6600 CONTINUE
  6700 CONTINUE
  6800 I=1,NOA
  DO 6900 J=1,NOT
    CALL BMVAL (NOT, RAD(1),M, RAD(J), A2(1,1), B2(1,1), C2(1,1),
    1 GRADR(I,J) = XDERV1
  6900 CONTINUE
  6901 RADR(1) = RAD(1)
  JN = 1
  GO TO 7200
  IF JN = JN+1
  7000 IF ( JN .LE. 12 ) GO TO 7010
  CALL CHECK ( 3 )
C

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7010 RETURN
7010 IF ( RADR(JN) = RADR(JN-1)+DELPL
7010 IF ( RADR(JN) .LE. RADR(NOT) ) GO TO 7200
7200 DO 7300 J=1,NOA
7200 VALU = RADR(JN)
7200 CALL BMVAL (NOT, RAD(1), M, VALU, A2(1,1), 32(1,1), C2(1,1),
1 PRES(J,JN) = 7
7300 CONTINUE
7300 IF ( RADR(JN) .LT. RAD(NOT) ) GO TO 7000
7300 NA = 0
7300 ANGFI(1) = ANG(1)
7300 JP = 1
7300 JQ = 1
7300 GO TO 7350
7340 JQ = JQ+1
7340 IF (JQ CHECK (4) .LE. 120 ) GO TO 7345
7340 CALL CHECK (4)
7340 RETURN
7345 ANGFI(JQ) = ANGFI(JQ-1) +DELAN
7350 IF (ANGFI(JQ)-ANG(NOA)) 7370,7370,7360
7360 ANGFI(JQ)=ANG(NOA)
7370 IF (ANGFI(JQ)-ANG(JP)) 7400,7390,7380
7380 JP = JP+1
7390 IF (JP-NOA) 7370,7370,7400
7390 NA = 1
7400 GO TO 7410
7400 NA = 0
7400 IF (JP CHECK (5) .LE. 15 ) GO TO 7405
7400 CALL CHECK (5)
7400 RETURN
7405 PERC = (ANGFI(JQ)- ANG(JP-1)) / (ANG(JP)-ANG(JP-1))
7410 DO 7440 J7=1,JN
7410 IF (NA) 7430,7420,7430
7420 PRESS (JQ,JZ) = (PRES(JP,JZ) -PRES(JP-1,J7)) * PERC+ PRES(JP-1,J7)
7420 GO TO 7440
7430 PRESS(JQ,J7) = PRES(JP,J7)
7440 CONTINUE
7440 IF ( ANGFI(JQ) .LT. ANG(NOA) ) GO TO 7340
7440 CALL CONPUN ( JN, RADP, JQ, ITEST )
7440 IF ( KNOT1 .EQ. 0 ) GO TO 7460
1 IF ( KNOT1 .EQ. 0 ) GO TO 7460
NOT = 3

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RAD(3) = XPPSAZ
RAD(2) = XPPSA2
DO 7451 I=1,NCA
  PTOP3(I,2) = XPPSAV(I,2)
  PTOP3(I,3) = XPPSAV(I,3)
7451 IF (ITEST.EQ.2) GO TO 9000
C CALCULATE CIRCUMFERENTIAL PRESSURE GRADIENTS
C
7460 IANG = 0
IFI = 0
IANG = IANG + 1
8000 IFI = IFI + 1
8001 IF (ANG(IANG) - ANGFI(IFI) ) 8010, 8002, 8001
8002 IF (IFI.EQ.1) GO TO 8010
ISKIP = 2
IFIP1 = IFI + 1
SKIP = 2.0
IFIM1 = IFI - 1
8003 DO 8006 I=IFIM1, IFIP1, ISKIP
DO 8004 J=1,JN
  PRP(J) = PRESS(I,J)
8004 CONTINUE
CALL RMFIT (5,JN,RADR(1),PRP(1),0.0,0.0,A2(1,1),B2(1,1),C2(1,1),D2(1,1))
DO 8005 J=1,NOT
CALL RMVAL (JN,RADR(1),M,RAD(J),A2(1,1),B2(1,1),C2(1,1),
1 PRESS(I,J) = 7
CONTINUE
DO 8007 J=1,NOT
  GRADTH(IANG,J) = (PRESS(IFIP1,J)-PRESS(IFIM1,J)) / (SKIP*DELTH*
1 RAD(J))
8006 CONTINUE
8007 IF ( IANG .LT. NOPAKS) GO TO 8000
GO TO 9000
8010 IFIP1 = IFI + 1
IFIM1 = IFI
ISKIP = 1
SKIP = 1.0
8000 GO TO 8003
END
SURROUTINE PRINT ( XPS, XPL, NISMAX, NISRAPI)

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SCORE0060
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00 2 J = 1,NPR
    IGOOD(I,J) = 0
    K = I * NPR + J - NPR
    IF (DAI(K).EQ.0.0) IGOOD(I,J) = 2
    2 CONTINUE
    KFAILS=0
    10 MNL=1
    AVE=0
    CNT=0
    DO 9 I=1,NR
    DO 9 J=1,NPR
    IF (IGOOD(I,J).GE.1) GO TO 9
    K = I * NPR + J - NPR
    AVE=AVE+DAI(K)
    CNT=CNT+1
    9 CONTINUE
    AVE=AVE/CNT
    SIGMA=0
    DO 11 I=1,NR
    DO 11 J=1,NPR
    IF (IGOOD(I,J).GE.1) GO TO 11
    K = I * NPR + J - NPR
    SIGMA=SIGMA+(DAI(K)-AVE)**2
    11 CONTINUE
    SIGMA=SQRT(SIGMA/(CNT-1.)) * 4.
    DEL=0
    M=0
    IF (MNL.EQ.2) GO TO 15
    MNL=2
    TAV=AVE
    15 DO 12 I=1,NR
    DO 12 J=1,NPR
    IF (IGOOD(I,J).GE.1) GO TO 12
    K = I * NPR + J - NPR
    DEL=ARS(AVE-DAI(K))
    IF (DELL.LE. SIGMA) GO TO 12
    IF (DELL.LT.DEL) GO TO 12
    DEL=DELL
    M=I
    N=J
    12 CONTINUE
    IF (M.EQ.0) GO TO 13
    IGOOD(M,N)=1
  
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13  GO TO 10
    DO 16 I=1,NR
    DO 16 J=1,NPR
    IF (IGOOD(I,J).NE.1) GO TO 16
    K = I * NPR + J - NPR
    IF (ABS(1.-DAI(K)/TAV) .GE.0.1) IGOOD(I,J)=2
16  CONTINUE
    DO 17 I = 1, NR
    DO 17 J = 1, NPR
    SUM=0
    COUNT=0
    K = I * NPR + J - NPR
    L = J * 10 + I - 10
    IFLAG(L) = IRLNK
    IF (IGOOD(I,J).LT.1) GO TO 17
    IFLAG(L) = ISTAR
    IF (J.EQ.1) GO TO 18
    IF (J.EQ.NPR) GO TO 18
    IF (IGOOD(I,J-1).NE.0) GO TO 18
    IF (IGOOD(I,J+1).NE.0) GO TO 18
    SUM=SUM+(DAI(K-1)+DAI(K+1))/2.
    COUNT=1
18  N=I-I/NR*NR+1
    M=I+NR-2-(I+NR-2)/NR*NR+1
    IF (IGOOD(N,J).NE.0) GO TO 19
    IF (IGOOD(M,J).NE.0) GO TO 19
    N = N * NPR + J - NPR
    M = M * NPR + J - NPR
    SUM=SUM+(DAI(N)+DAI(M))/2.
    COUNT=COUNT+1
19  IF (COUNT.EQ.0) COUNT=1.
    IF (DAI(K)=SUM/COUNT)
    IF (DAI(K).EQ.0.0) IFLAG(L) = IPLUS
    IF (DAI(K).EQ.0.0 .AND. J .EQ. 1) DAI(K) = DAI(K+1)
    IF (DAI(K).EQ.0.0 .AND. J .EQ. NPR) DAI(K) = DAI(K-1)
    IF (DAI(K).EQ.0.0) KFAILS=1
    IF (DAI(K).NE.0.0) IGOOD(I,J)=0
17  CONTINUE
    IF (KFAILS .EQ. 0 .OR. ICYCLE .EQ. 1) GO TO 99
    KFAILS = 0
    ICYCLE = 1
    DO 22 I = 1, 50
    IF (IGOOD(I) = IFLAG(I)
22  GO TO 20
  
```

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99 IF ( ICYCLE .EQ. 0 ) RETURN
DO 23 I = 1, 50
IF ( IFLAG(I) .NE. IBLNK ) IFLAG(I) = IMINUS
23 IFLAG(I) = IFLAG(I)
RETURN
END
SUBROUTINE SORT1 ( JS, X, Y, JTOT )
DIMENSION JS ( 1 ), X ( 1 ), Y ( 1 )
IMAX = JTOT - 1
DO 2 I = 1, IMAX
IP1 = I + 1
DO 2 J = IP1, JTOT
IF ( X(I) - X(J) )
X ( J ) = X ( I )
X ( I ) = X ( J )
Y ( J ) = Y ( I )
Y ( I ) = Y ( J )
L = JS ( J )
JS ( J ) = JS ( I )
JS ( I ) = L
CONTINUE
RETURN
END
SUBROUTINE SPLSD3 ( X, Y, N, C )
DIMENSION X(1), Y(1), C(1)
T = 1
H = X(2) - X(1)
GO TO 20
10 HX = X(I+1) - X(I)
20 HX = - H/HX
C(I) = ( ( 1.0 - ZE ) * Y(I) + 7E * Y(I+1) - Y(I-1) ) / ( FK * FK )
ZE = 1.0 - ZE
IF ( I .LT. N ) GO TO 10
C(I) = 0.0
C(1) = 0.0
RETURN
END
SUBROUTINE SPREAD ( TIME, DATA )
DIMENSION DATA(1)

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SCPE0040
 SCPE0045
 SCPE0050
 SCPE0055
 SCPE0060
 SCPE0065
 SCPE0070
 SCPE0075
 SCPE0080
 SCPE0085
 SCPE0090
 SCPE0095
 SCPE0100
 SCPE0105
 SCPE0110
 SCPE0115
 SCPE0120
 SCPE0125
 SCPE0130
 SCPE0135
 SCPE0140
 SCPE0145
 SCPE0150
 SCPE0155
 SCPE0160
 SCPE0165
 SCPE0170
 SCPE0175
 SCPE0180
 SCPE0185
 SCPE0190
 SCPE0195
 SCPE0200
 SCPE0205
 SCPE0210
 SCPE0215
 SCPE0220
 SCPE0225
 SCPE0230
 SCPE0235
 SCPE0240
 SCPE0245
 SCPE0250
 SCPE0255
 SCPE0260
 SCPE0265
 SCPE0270
 SCPE0275
 SCPE0280
 SCPE0285
 SCPE0290
 SCPE0295
 SCPE0300

C THIS SPACE IS ALLOCATED FOR A TAPE READING ROUTINE
C
RETURN
END

SPRE0040
SPRE0050
SPRE0060
SPRE0070

ME-13 DATA POINT 24 STEADY STATE

* VASTICITY MAP PROGRAM
 A PRODUCT OF
 GGLV. ENGINEERING (457/NJSEA)

*** CONTROL CASE INPUT ***

IP-100	7	YSCAL-EN	0	DELVS	0.04	DELVOR	6.00.00	RIN	5.65
ROT	17.33	DOT	6	NDRAKS	3	DELAN	2.00		
ROT	17.33	T2	661.00	ALAFLO	200.200	DELDT1	2		
ANGLES	18.00	AS.00	30.00			135.00			
18.00		270.00	270.00			115.00			
22.00		9.9190	11.9220			13.7560			
12.3100		13.7560							

44-18 DATA POINT 74 T-REV STATE

CAS NUMBER = 1.00000
 INPUT RADIIUS / OUTER RADIIUS = .32436

NO. OF RAKES = 4

NO. OF PROBES PER RAKE = 6

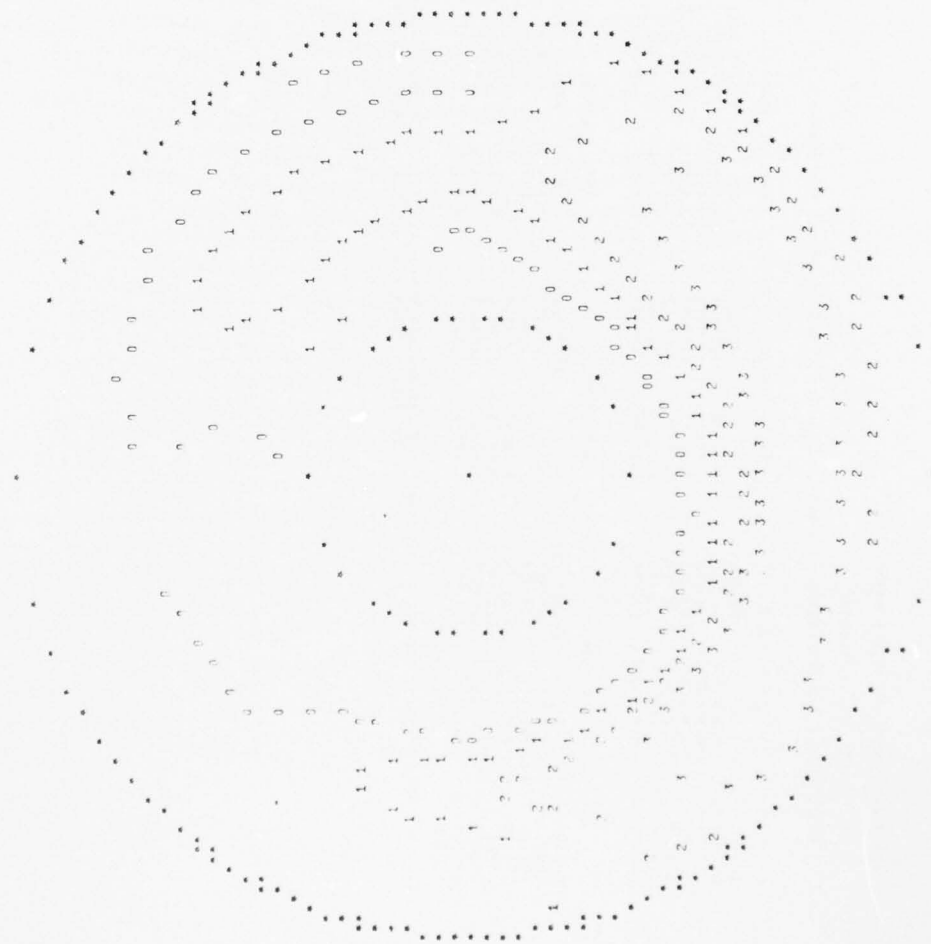
RAKE = 0.0			RAKE = 45.0			RAKE = 90.0			RAKE = 135.0		
PTL	PTL/PTLAVG	PTL	PTL/PTLAVG	PTL	PTL/PTLAVG	PTL	PTL/PTLAVG	PTL	PTL/PTLAVG	PTL	PTL/PTLAVG
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000

RAKE = 180.0			RAKE = 225.0			RAKE = 270.0			RAKE = 315.0		
PTL	PTL/PTLAVG	PTL	PTL/PTLAVG	PTL	PTL/PTLAVG	PTL	PTL/PTLAVG	PTL	PTL/PTLAVG	PTL	PTL/PTLAVG
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000
3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000	3.1073	1.0000

AREA WEIGHTED ON SOURCE AVERAGE = 0.2349 PTL/PTLAVG MINIMUM = .93173 DISTORTION (MAX-MIN)/AVG = .1706
 PTL/PTLAVG MAXIMUM = 1.11585 DISTORTION (AVG-MIN)/AVG = .0513

WATER POINT 24 ST 414 ST

Area: 400.00
Time: 1.0000
SYMBOL (LUNING APT) PT/PTLAVG
0
1
2
3



MAIN DATA POINT 24 STEADY STATE

CAS NUMBER = 1.00000

*** RADIAL VORTICITY ***

RAX =	0.0	50.0	90.0	135.0	180.0	225.0	270.0	315.0
RY / OY								
.4183	72.12	-51.33	-199.70	.36	.23	5.27	12.03	35.42
.5704	24.22	42.07	78.41	58.24	-104.31	-93.62	-51.03	-31.97
.6301	109.13	72.64	78.89	93.43	-2.54	-63.03	-146.17	-121.53
.7317	110.14	29.83	145.35	94.67	-2.98	-46.11	-143.29	-78.13
.8410	23.03	10.90	162.66	57.76	2.50	-37.47	-116.63	-71.51
.9526	-7.54	46.27	51.99	42.47	39.55	-43.73	-54.07	-23.26

*** CIRCUMFERENTIAL VORTICITY ***

RAX =	0.0	45.0	90.0	135.0	180.0	225.0	270.0	315.0
RY / OY								
.4183	371.73	262.03	126.07	-1814.22	-831.64	621.43	732.25	-154.29
.5704	-156.06	-288.73	-782.73	-510.08	-1029.61	-124.40	-1156.69	33.53
.6301	1.71	166.03	109.18	-214.11	-378.64	-567.40	84.54	-316.90
.7317	119.14	613.75	761.61	-11.11	609.55	26.67	967.01	-23.40
.8410	-20.00	464.69	-154.05	535.48	586.63	276.16	-141.86	43.89
.9526	-64.53	61.12	-1007.96	2245.04	-509.65	921.42	373.69	-104.04

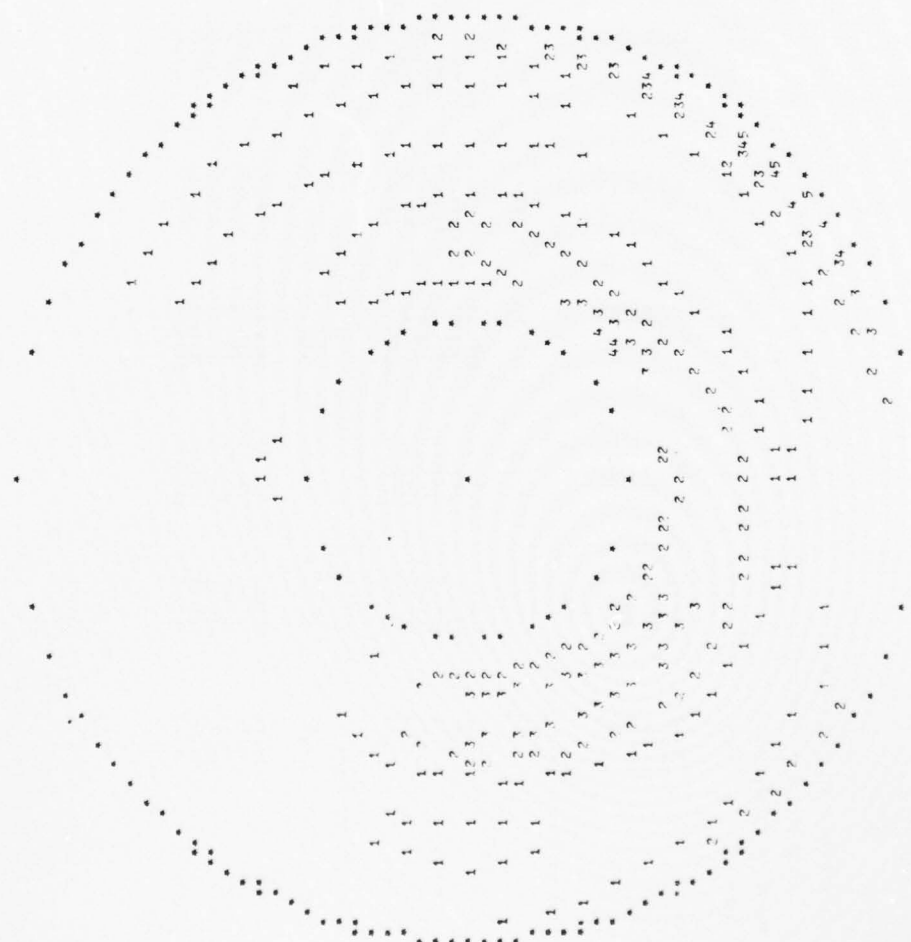
*** TOTAL VORTICITY ***

RAX =	0.0	45.0	90.0	135.0	180.0	225.0	270.0	315.0
RY / OY								
.4183	460.72	263.96	222.50	1814.22	431.64	421.45	732.34	162.20
.5704	280.42	237.43	786.55	513.19	1034.83	1347.62	1154.02	63.94
.6301	190.93	191.70	136.11	223.12	178.65	571.59	170.56	353.15
.7317	192.72	614.73	777.76	124.67	609.66	92.05	544.90	241.11
.8410	20.21	464.83	227.96	533.77	586.63	274.69	131.65	53.08
.9526	46.73	69.15	1003.29	2245.45	601.16	922.45	379.14	108.39

HA-18 DATA POINT 24 STEADY STATE

AV.VORTICITY	485.29389
TIME	1.00000
ITER	593.00000
PSI	3.72060
ALPHA	200.70000

SUBROUTINE	SYMBOL (LOOKING AFT) VORTICITY
0	0
1	1
2	2
3	3
4	4
5	5



Appendix B

DEPARTMENT OF THE AIR FORCE
HEADQUARTERS AERONAUTICAL SYSTEMS DIVISION (AFSC)
WRIGHT-PATTERSON AIR FORCE BASE, OHIO 45433



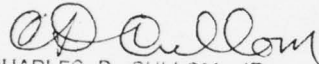
REPLY TO
ATTN OF: YFEJ (Mr. Kanouse/53043)

20 APR 1976

SUBJECT: Distortion Report

TO: ASD/ENFPA (S. Clark)

YFEJ has reviewed the subject report and have determined it to be classified properly and have no objection to its publication.


CHARLES D. CULLOM, JR.
Chief, Propulsion & Power Division
Deputy for F-15/JEPO

1 Atch
Distortion Report

